

Dynamics of an adaptive vibration absorber

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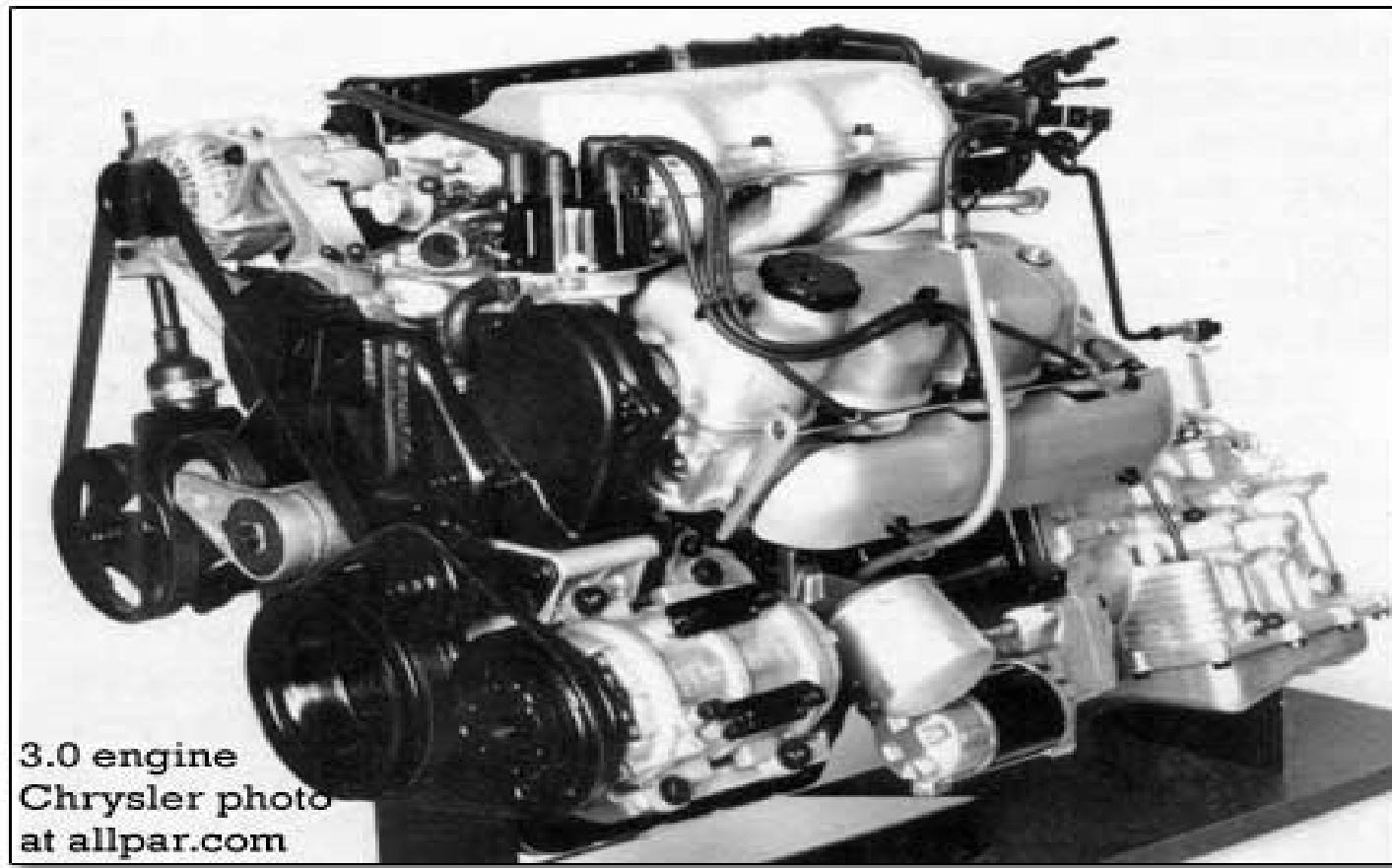
Outline

- Motivation
 - Design
 - Mathematical model
 - Adaptive configurations
 - Stability
 - Adaptability
 - Future work
-

Motivation

- Vibrations generated by reciprocating machinery, such as, for example, internal combustion engines or piston type compressors.
-

Motivation

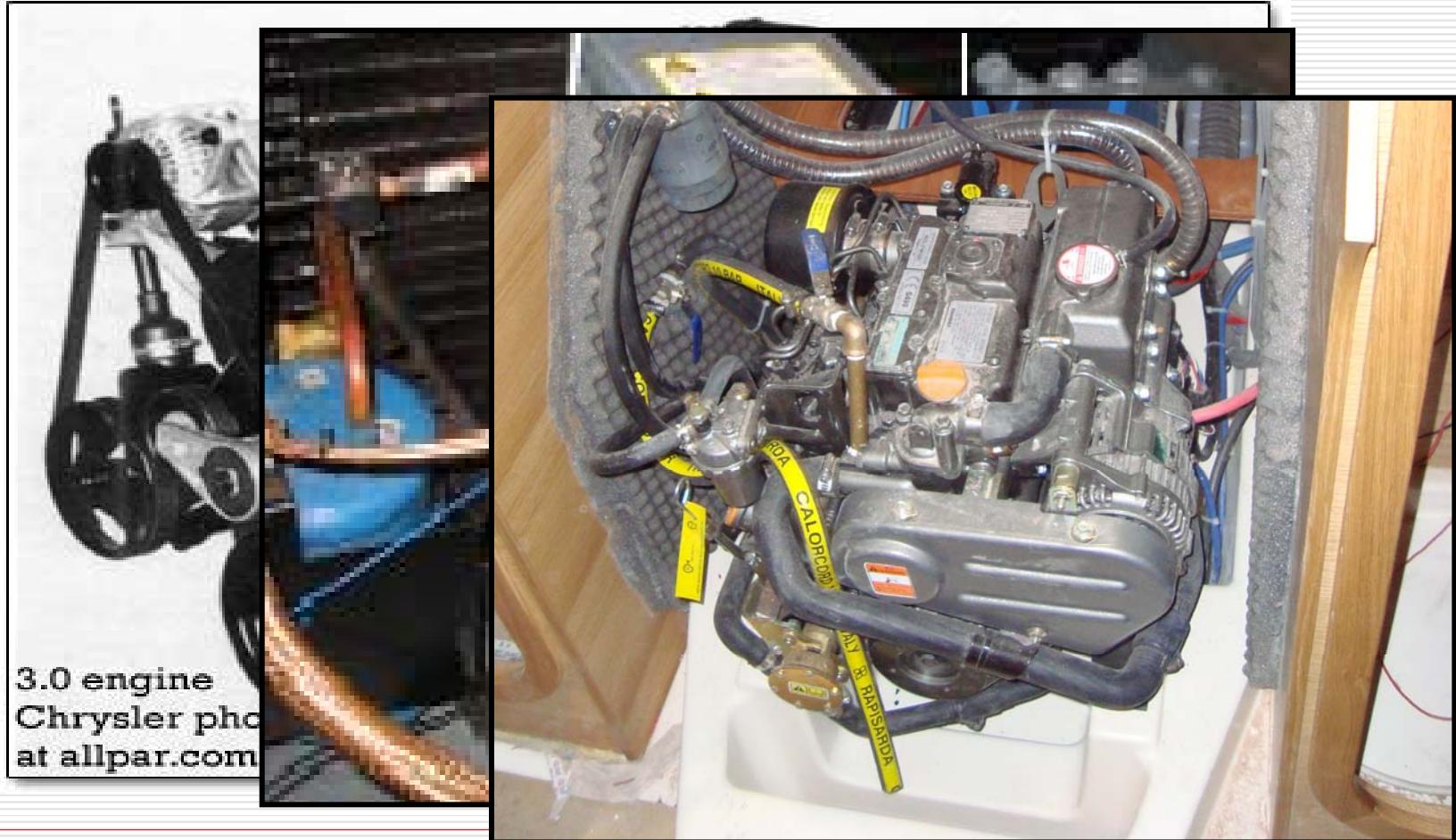


3.0 engine
Chrysler photo
at allpar.com

Motivation



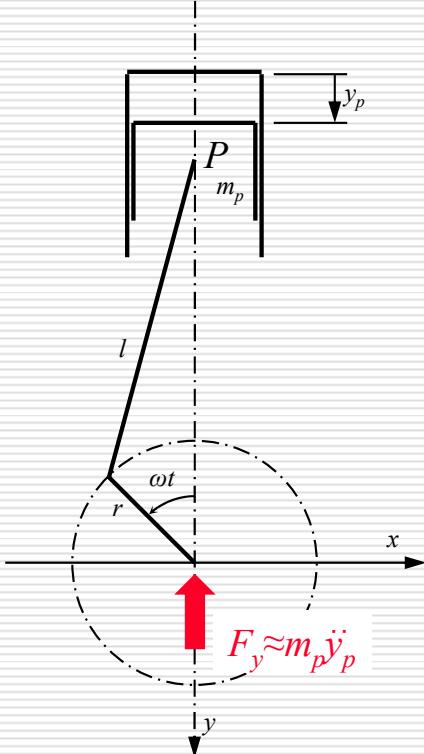
Motivation



Motivation

- Such machines generate vibrations at discrete frequencies: multiples of the machine RPM.
- These multiples are often called **ENGINE ORDERS.**

Motivation



$$y_p = r(1 - \cos \omega t) + l \left(1 - \sqrt{1 - \left(\frac{r}{l} \right)^2 \sin^2 \omega t} \right)$$

$$\frac{l}{r} = \alpha$$

$$\frac{y_p}{r} = 1 - \cos \omega t + \alpha - \alpha \sqrt{1 - \frac{\sin^2 \omega t}{\alpha^2}}$$

Taylor's series expansion; up to the 7th order

$$\frac{y_p}{r} = 1 - \cos(\omega t) + \frac{1}{2\alpha} \sin^2(\omega t) + \frac{1}{8\alpha^3} \sin^4(\omega t) + \frac{1}{16\alpha^5} \sin^6(\omega t) + O\left(\frac{1}{\alpha^7}\right)$$

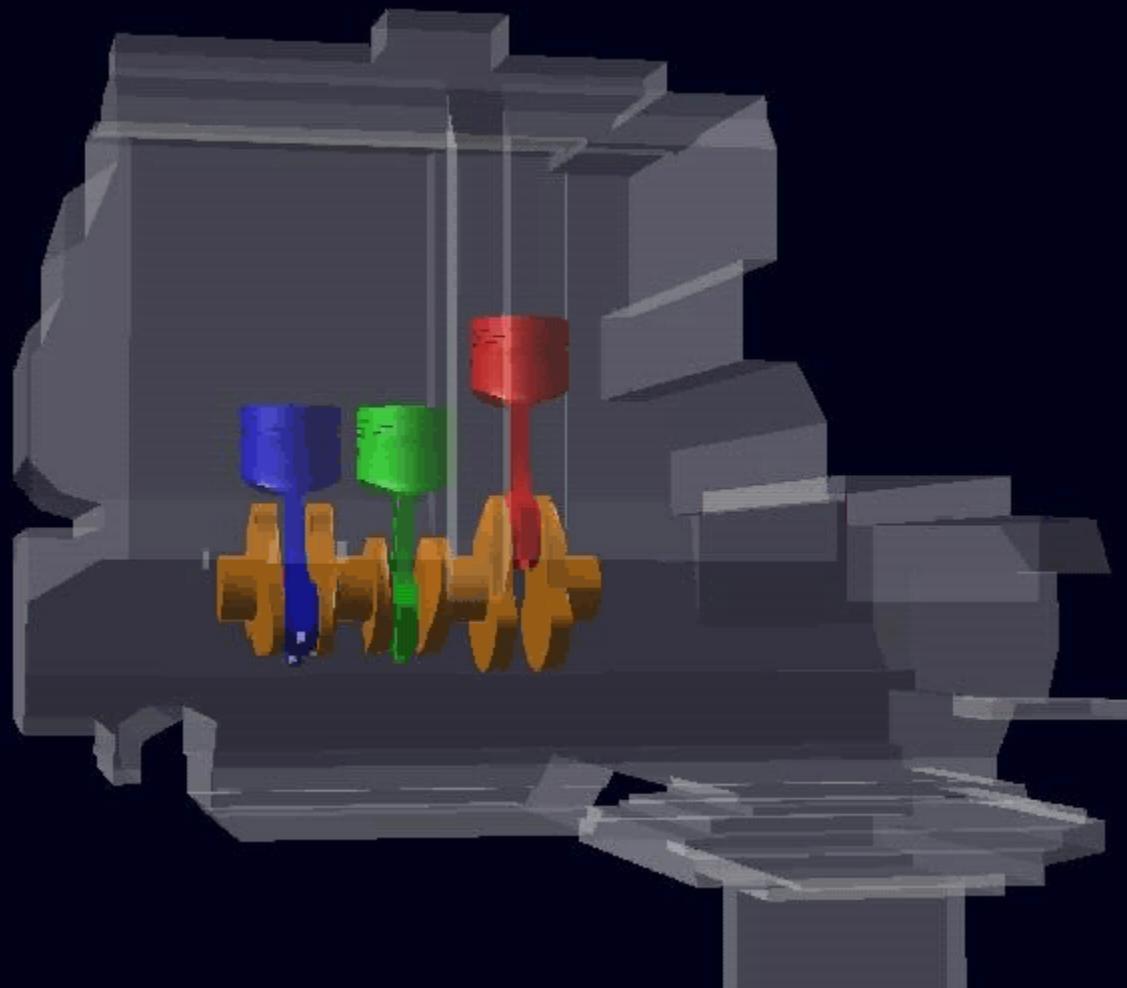
$$\sin^2 \alpha = \frac{1 - \cos 2\alpha}{2}; \frac{d^2}{dt^2}$$

$$\frac{\ddot{y}_p}{r} = \omega^2 \left[\cos \omega t + \frac{128\alpha^4 + 32\alpha^2 + 15}{128\alpha^5} \cos(2\omega t) - \frac{8\alpha^2 + 6}{32\alpha^5} \cos(4\omega t) + \frac{9}{128\alpha^5} \cos(6\omega t) \right]$$

"ENGINE ORDERS"

Motivation

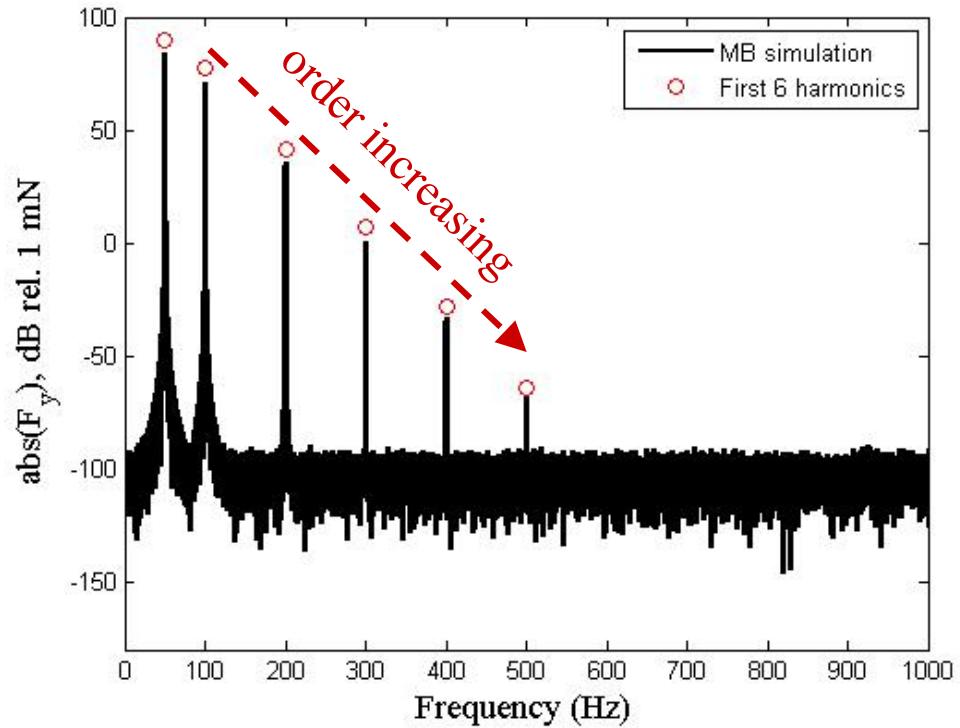
Last_Run Time= 0.0000 Frame=1



Motivation

- Lower orders are more important because the vibration amplitude quickly decreases with increase of the engine order...

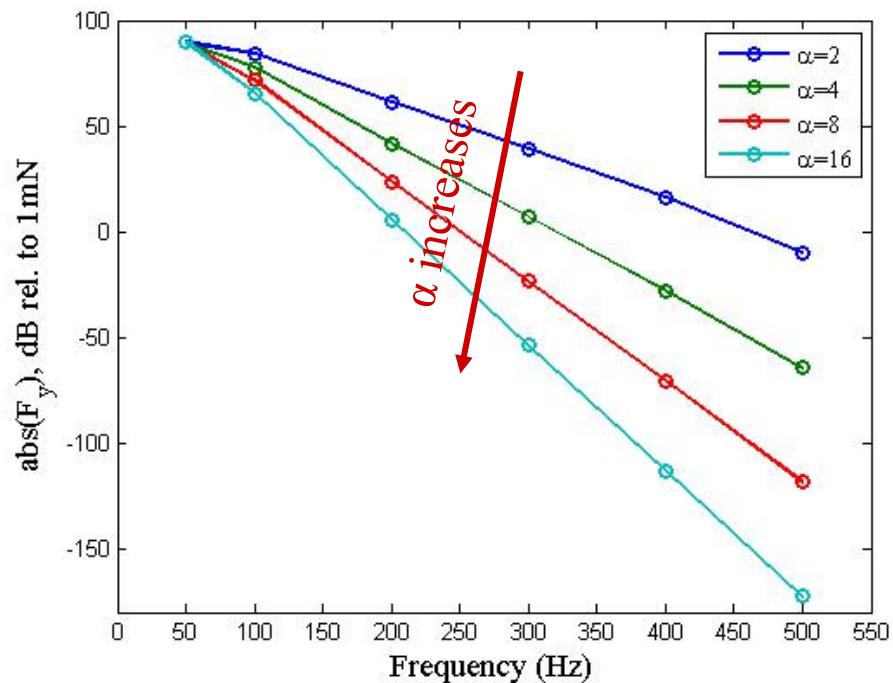
$$N = 3000 \text{ min}^{-1}; \frac{l}{r} = \alpha = 4$$



Motivation

- ...especially for relatively long connecting rods.

$$\alpha = \frac{l}{r}$$



Motivation

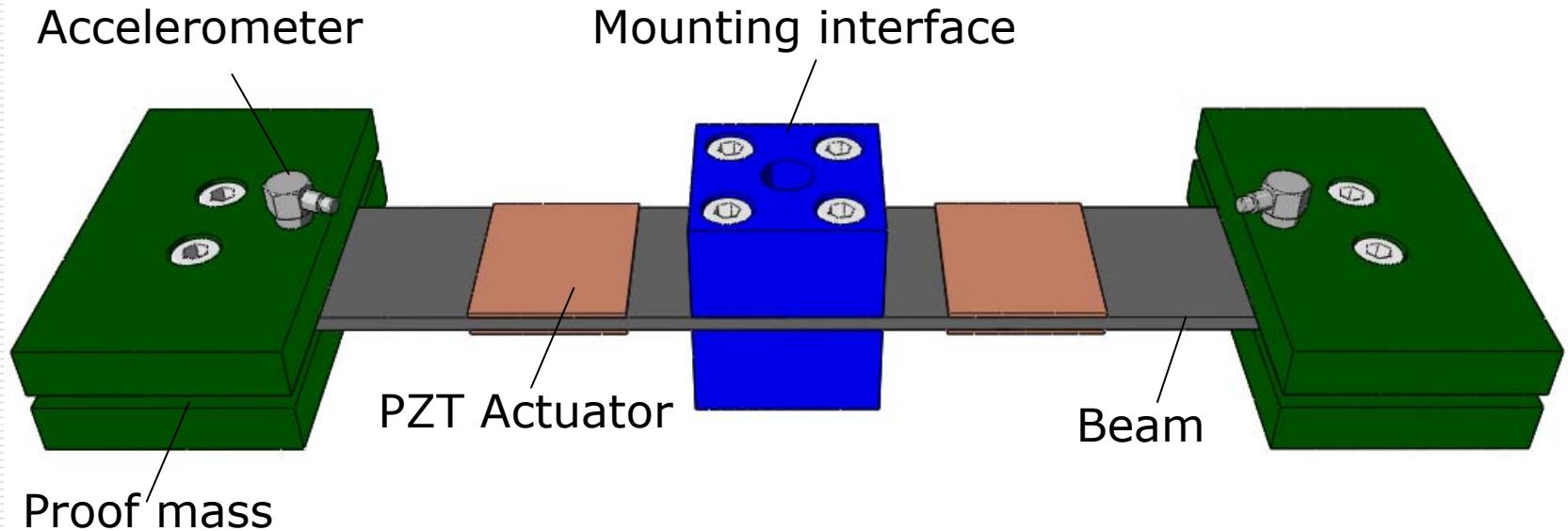
- A possible solution might be an adaptive vibration absorber.
 - Such devices efficiently absorb vibrations at their resonant frequency.
 - The resonant frequency should be tunable so as to follow the machine RPM.
-

Design

- Simple construction
 - Low cost
 - Little space required at the mounting point
-

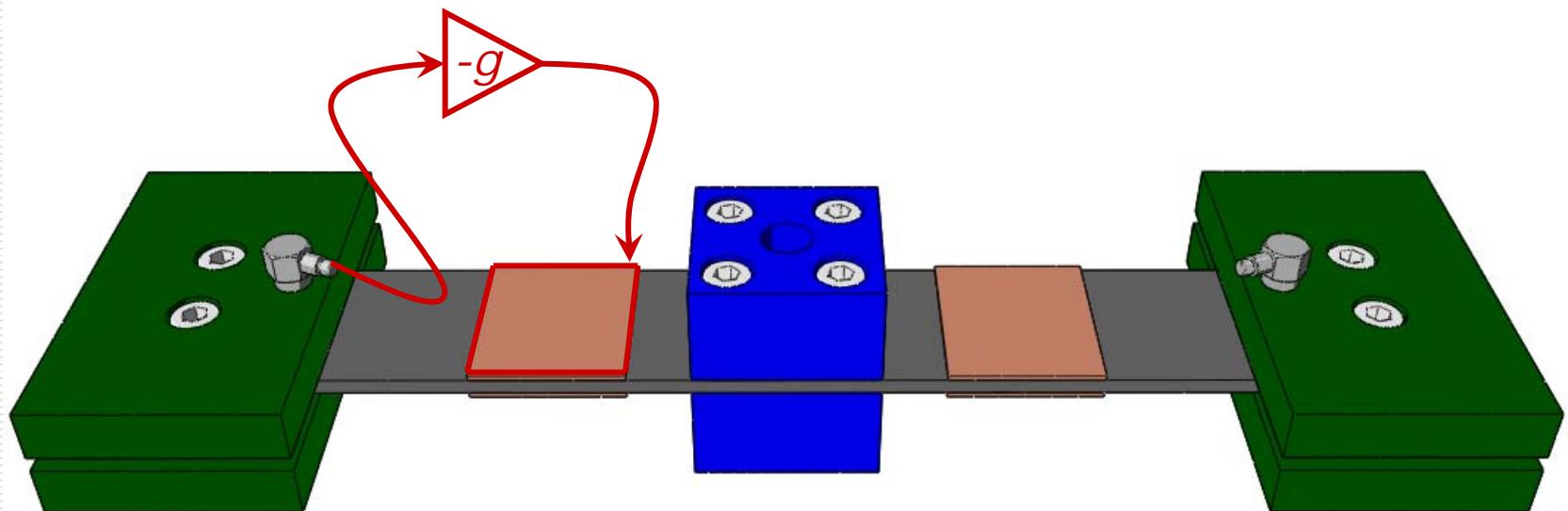
Design

□ Beam type vibration absorber



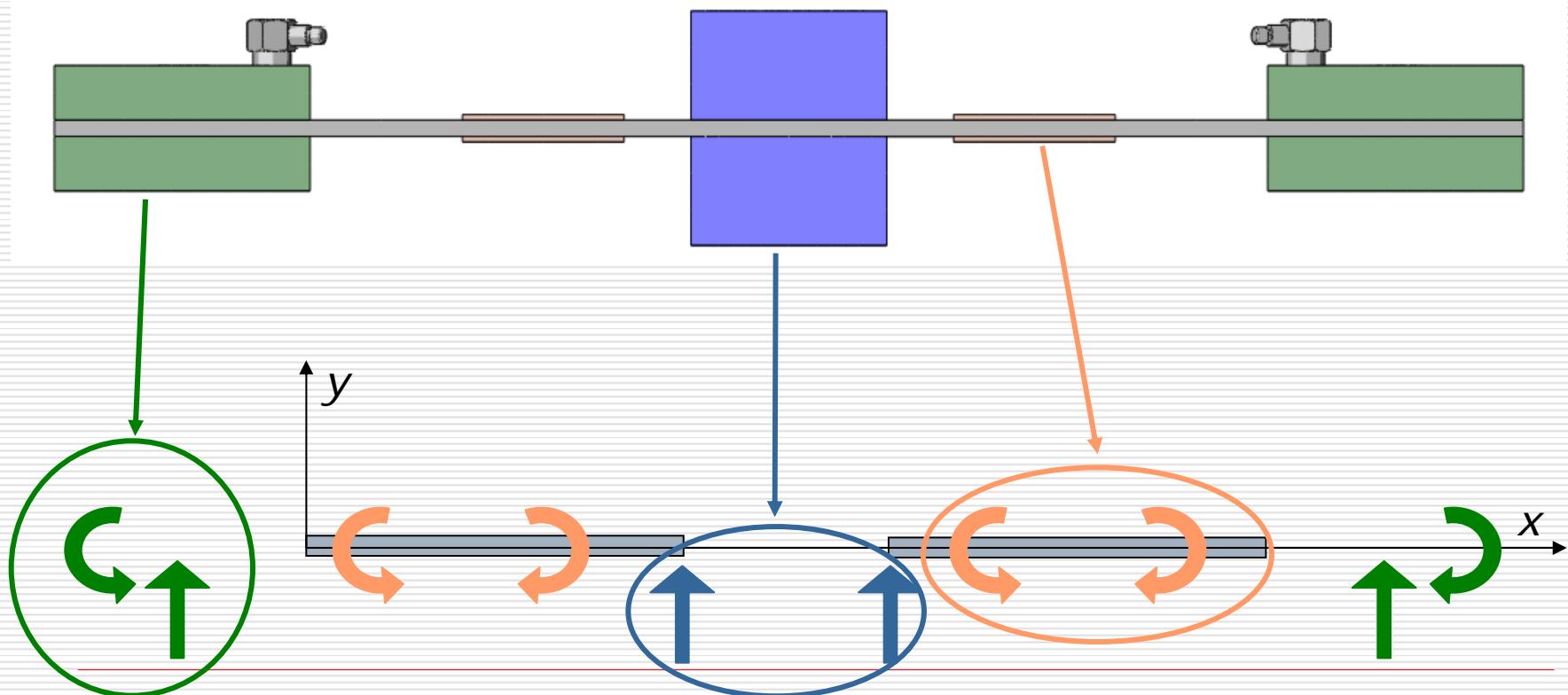
Design

- ☐ Negative acceleration feedback – active mass



Mathematical model

□ Mobility – impedance approach



Mathematical model: Formulation

$$\mathbf{F}_{\text{active}} = -j\omega \mathbf{G}^{\frac{\mathbf{F}' = \mathbf{Z}\mathbf{V}}{\mathbf{V}}} \mathbf{V}_{\text{active}}$$

$$\mathbf{V} = \mathbf{Y}\mathbf{F} + (\mathbf{I} + j\omega \mathbf{Q}\mathbf{G})^{-1} = \mathbf{R}$$

$$\mathbf{Z} = \begin{bmatrix} \mathbf{Z}^{f,v} & \mathbf{Z}^{t,v} \\ \mathbf{Z}^{f,\alpha} & \mathbf{Z}^{t,\alpha} \end{bmatrix}$$

$$\begin{bmatrix} \mathbf{V} \\ \mathbf{a} \end{bmatrix} = \begin{bmatrix} \mathbf{Y}^{v,f} & \mathbf{Y}^{v,t} \\ \mathbf{Y}^{\alpha,f} & \mathbf{Y}^{\alpha,t} \end{bmatrix} \begin{bmatrix} \mathbf{f} \\ \mathbf{t} \end{bmatrix} + \begin{bmatrix} \mathbf{Y}^{v,f} & \mathbf{Y}^{v,t} \\ \mathbf{Y}^{\alpha,f} & \mathbf{Y}^{\alpha,t} \end{bmatrix} \begin{bmatrix} \mathbf{f}' \\ \mathbf{t}' \end{bmatrix}$$

$$\mathbf{V} - \mathbf{Y}\mathbf{Z}\mathbf{V} = \mathbf{Y}\mathbf{F}$$

$$\mathbf{V} = \mathbf{Q}\mathbf{F} \quad (\mathbf{I} - \mathbf{Y}\mathbf{Z})^{-1}\mathbf{Y} = \mathbf{Q}$$

$$\mathbf{V}_{\text{active}} + j\omega \mathbf{Q}\mathbf{G}\mathbf{V}_{\text{active}} = \mathbf{Q}\mathbf{F}$$

$$(\mathbf{I} - \mathbf{Y}\mathbf{Z})\mathbf{V} = \mathbf{Y}\mathbf{F}$$

$$\mathbf{V} = \mathbf{Y}\mathbf{F} + \mathbf{Y}\mathbf{Z}\mathbf{V}$$

$$\mathbf{V}_{\text{active}} = \mathbf{R}\mathbf{Q}\mathbf{F}$$

$$\mathbf{R}\mathbf{Q} = \mathbf{T}$$

$$(\mathbf{I} + j\omega \mathbf{Q}\mathbf{G})\mathbf{V}_{\text{active}} = \mathbf{Q}\mathbf{F}$$

$$\mathbf{V}_{\text{active}} = \mathbf{Q}\mathbf{F} + \mathbf{Q}\mathbf{F}_{\text{active}}$$

$$\mathbf{V}_{\text{active}} = \mathbf{T}\mathbf{F}$$

$$\mathbf{V}_{\text{active}} = (\mathbf{I} + j\omega \mathbf{Q}\mathbf{G})^{-1}\mathbf{Q}\mathbf{F}$$

Mathematical model: Mobility matrices

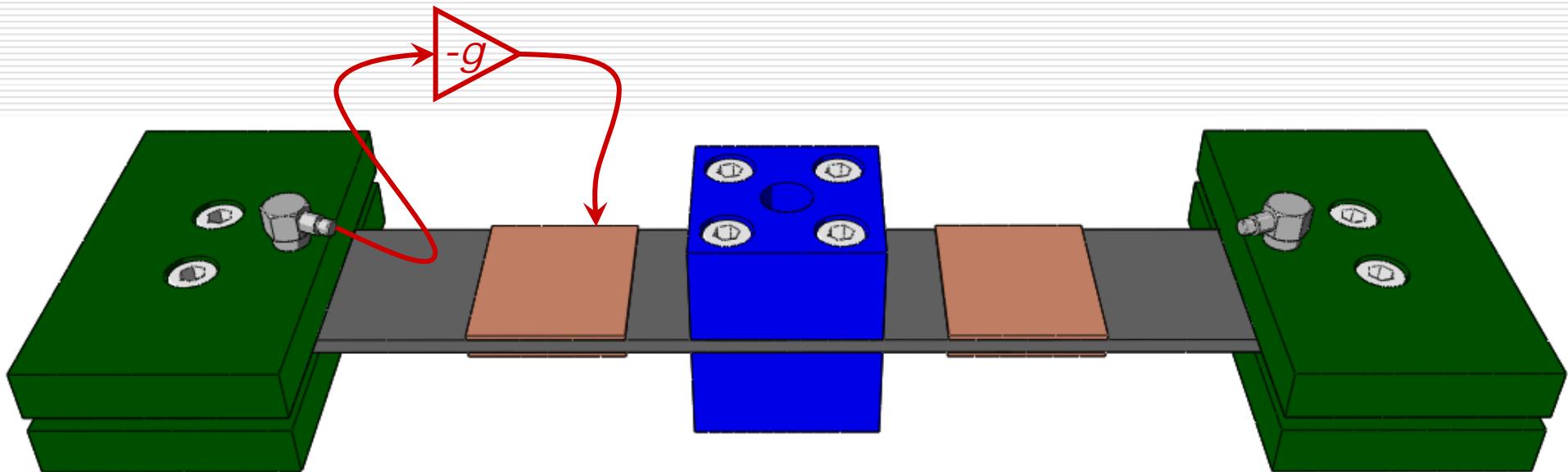
$$\mathbf{Y}^{v,f} = \begin{bmatrix} Y_{m1,m1}^{v,f} & Y_{m1,m2}^{v,f} & Y_{m1,m3}^{v,f} & Y_{m1,c1}^{v,f} & Y_{m1,c2}^{v,f} & \cdots & Y_{m1,cM}^{v,f} & Y_{m1,e1}^{v,f} & Y_{m1,e2}^{v,f} & \cdots & Y_{m1,eN}^{v,f} \\ Y_{m2,m1}^{v,f} & Y_{m2,m2}^{v,f} & Y_{m2,m3}^{v,f} & Y_{m2,c1}^{v,f} & Y_{m2,c2}^{v,f} & \cdots & Y_{m2,cM}^{v,f} & Y_{m2,e1}^{v,f} & Y_{m2,e2}^{v,f} & \cdots & Y_{m2,eN}^{v,f} \\ Y_{m3,m1}^{v,f} & Y_{m3,m2}^{v,f} & Y_{m3,m3}^{v,f} & Y_{m3,c1}^{v,f} & Y_{m3,c2}^{v,f} & \cdots & Y_{m3,cM}^{v,f} & Y_{m3,e1}^{v,f} & Y_{m3,e2}^{v,f} & \cdots & Y_{m3,eN}^{v,f} \\ Y_{cl,m1}^{v,f} & Y_{cl,m2}^{v,f} & Y_{cl,m3}^{v,f} & Y_{cl,c1}^{v,f} & Y_{cl,c2}^{v,f} & \cdots & Y_{cl,cM}^{v,f} & Y_{cl,e1}^{v,f} & Y_{cl,e2}^{v,f} & \cdots & Y_{cl,eN}^{v,f} \\ Y_{c2,m1}^{v,f} & Y_{c2,m2}^{v,f} & Y_{c2,m3}^{v,f} & Y_{c2,c1}^{v,f} & Y_{c2,c2}^{v,f} & \cdots & Y_{c2,cM}^{v,f} & Y_{c2,e1}^{v,f} & Y_{c2,e2}^{v,f} & \cdots & Y_{c2,eN}^{v,f} \\ \vdots & \vdots & \vdots & \vdots & \vdots & \cdots & \vdots & \vdots & \vdots & \cdots & \vdots \\ Y_{cM,m1}^{v,f} & Y_{cM,m2}^{v,f} & Y_{cM,m3}^{v,f} & Y_{cM,c1}^{v,f} & Y_{cM,c2}^{v,f} & \cdots & Y_{cM,cM}^{v,f} & Y_{cM,e1}^{v,f} & Y_{cM,e2}^{v,f} & \cdots & Y_{cM,eN}^{v,f} \\ Y_{el,m1}^{v,f} & Y_{el,m2}^{v,f} & Y_{el,m3}^{v,f} & Y_{el,c1}^{v,f} & Y_{el,c2}^{v,f} & \cdots & Y_{el,cM}^{v,f} & Y_{el,e1}^{v,f} & Y_{el,e2}^{v,f} & \cdots & Y_{el,eN}^{v,f} \\ Y_{e2,m1}^{v,f} & Y_{e2,m2}^{v,f} & Y_{e2,m3}^{v,f} & Y_{e2,c1}^{v,f} & Y_{e2,c2}^{v,f} & \cdots & Y_{e2,cM}^{v,f} & Y_{e2,e1}^{v,f} & Y_{e2,e2}^{v,f} & \cdots & Y_{e2,eN}^{v,f} \\ \vdots & \vdots & \vdots & \vdots & \vdots & \cdots & \vdots & \vdots & \vdots & \cdots & \vdots \\ Y_{eN,m1}^{v,f} & Y_{eN,m2}^{v,f} & Y_{eN,m3}^{v,f} & Y_{eN,c1}^{v,f} & Y_{eN,c2}^{v,f} & \cdots & Y_{eN,cM}^{v,f} & Y_{eN,e1}^{v,f} & Y_{eN,e2}^{v,f} & \cdots & Y_{eN,eN}^{v,f} \\ Y_{el,m1}^{\alpha,t} & Y_{el,m2}^{\alpha,t} & Y_{el,m3}^{\alpha,t} & Y_{el,c1}^{\alpha,t} & Y_{el,c2}^{\alpha,t} & \cdots & Y_{el,cM}^{\alpha,t} & Y_{el,e1}^{\alpha,t} & Y_{el,e2}^{\alpha,t} & \cdots & Y_{el,eN}^{\alpha,t} \\ Y_{e2,m1}^{\alpha,t} & Y_{e2,m2}^{\alpha,t} & Y_{e2,m3}^{\alpha,t} & Y_{e2,c1}^{\alpha,t} & Y_{e2,c2}^{\alpha,t} & \cdots & Y_{e2,cM}^{\alpha,t} & Y_{e2,e1}^{\alpha,t} & Y_{e2,e2}^{\alpha,t} & \cdots & Y_{e2,eN}^{\alpha,t} \\ \vdots & \vdots & \vdots & \vdots & \vdots & \cdots & \vdots & \vdots & \vdots & \cdots & \vdots \\ Y_{eN,m1}^{\alpha,t} & Y_{eN,m2}^{\alpha,t} & Y_{eN,m3}^{\alpha,t} & Y_{eN,c1}^{\alpha,t} & Y_{eN,c2}^{\alpha,t} & \cdots & Y_{eN,cM}^{\alpha,t} & Y_{eN,e1}^{\alpha,t} & Y_{eN,e2}^{\alpha,t} & \cdots & Y_{eN,eN}^{\alpha,t} \end{bmatrix}$$

Mathematical model: Impedance matrices

$$\begin{aligned}
 \mathbf{Z}^{f,v} = & \begin{bmatrix}
 Z_{m1,m1}^{f,v} & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\
 0 & Z_{m2,m2}^{f,v} & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\
 0 & 0 & Z_{m3,m3}^{f,v} & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\
 0 & 0 & 0 & Z_{c1,c1}^{f,v} & 0 & 0 & 0 & 0 & 0 & 0 \\
 0 & 0 & 0 & 0 & Z_{c2,c2}^{f,v} & 0 & 0 & 0 & 0 & 0 \\
 0 & 0 & 0 & 0 & 0 & \ddots & 0 & 0 & 0 & 0 \\
 0 & Z_{m1,m1}^{t,\alpha} & 0 & 0 & 0 & 0 & Z_{cM,cM}^{f,v} & Z_{e1,e1}^{f,v} & Z_{e2,e2}^{f,v} & Z_{eN,eN}^{f,v} \\
 0 & 0 & Z_{m2,m2}^{t,\alpha} & 0 & 0 & 0 & 0 & Z_{e1,e2}^{f,v} & Z_{e2,e1}^{f,v} & 0 \\
 0 & 0 & Z_{m3,m3}^{t,\alpha} & 0 & 0 & 0 & 0 & 0 & Z_{e2,e2}^{f,v} & Z_{eN,eN}^{f,v} \\
 0 & 0 & 0 & Z_{c1,c1}^{t,\alpha} & 0 & 0 & 0 & 0 & 0 & Z_{eN,eN}^{f,v} \\
 0 & 0 & 0 & 0 & Z_{c2,c2}^{t,\alpha} & 0 & 0 & 0 & 0 & Z_{eN,eN}^{f,v} \\
 0 & 0 & 0 & 0 & 0 & \ddots & 0 & 0 & 0 & 0 \\
 \end{bmatrix} \\
 \mathbf{Z}^{t,\theta} = & \begin{bmatrix}
 Z_{m1,m1}^{t,\theta} & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\
 0 & Z_{m2,m2}^{t,\theta} & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\
 0 & 0 & Z_{m3,m3}^{t,\theta} & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\
 0 & 0 & 0 & Z_{c1,c1}^{t,\theta} & 0 & 0 & 0 & 0 & 0 & 0 \\
 0 & 0 & 0 & 0 & Z_{c2,c2}^{t,\theta} & 0 & 0 & 0 & 0 & 0 \\
 0 & 0 & 0 & 0 & 0 & \ddots & 0 & 0 & 0 & 0 \\
 \end{bmatrix} \\
 \mathbf{Z}^{t,\alpha} = & \begin{bmatrix}
 Z_{m1,m1}^{t,\alpha} & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\
 0 & Z_{m2,m2}^{t,\alpha} & 0 & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\
 0 & 0 & Z_{m3,m3}^{t,\alpha} & 0 & 0 & 0 & 0 & 0 & 0 & 0 \\
 0 & 0 & 0 & Z_{c1,c1}^{t,\alpha} & 0 & 0 & 0 & 0 & 0 & 0 \\
 0 & 0 & 0 & 0 & Z_{c2,c2}^{t,\alpha} & 0 & 0 & 0 & 0 & 0 \\
 0 & 0 & 0 & 0 & 0 & \ddots & 0 & 0 & 0 & 0 \\
 \end{bmatrix}
 \end{aligned}$$

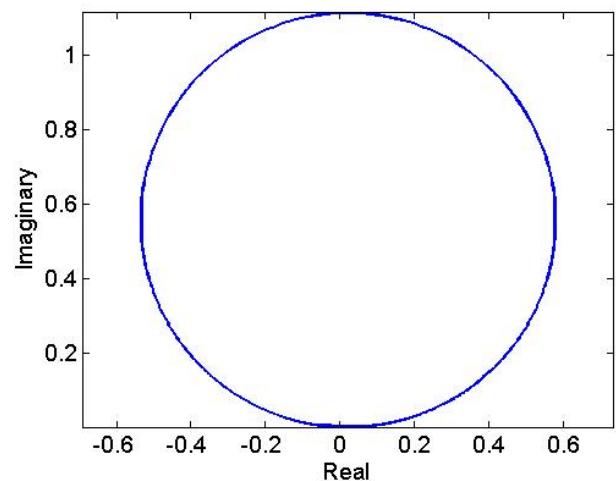
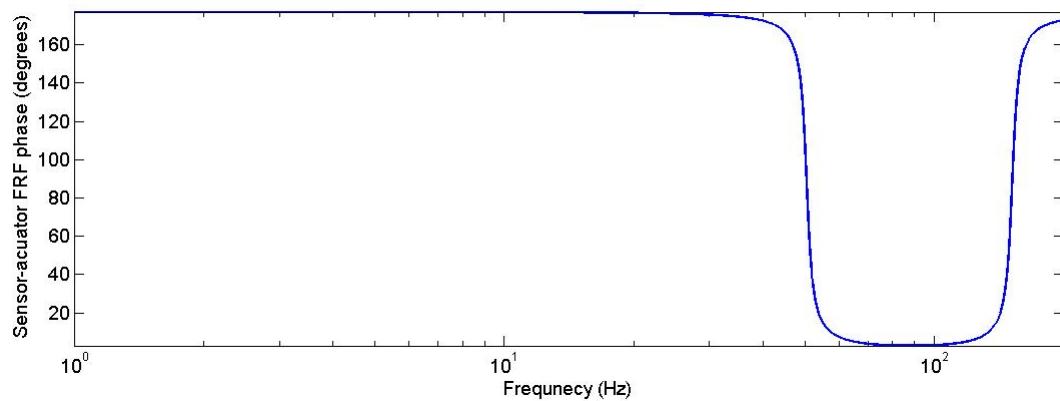
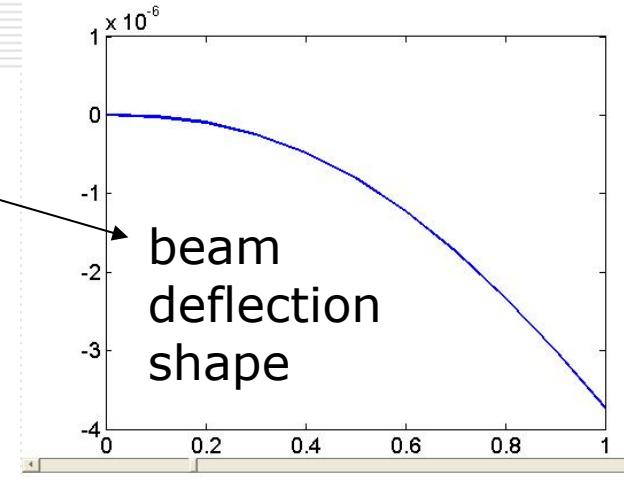
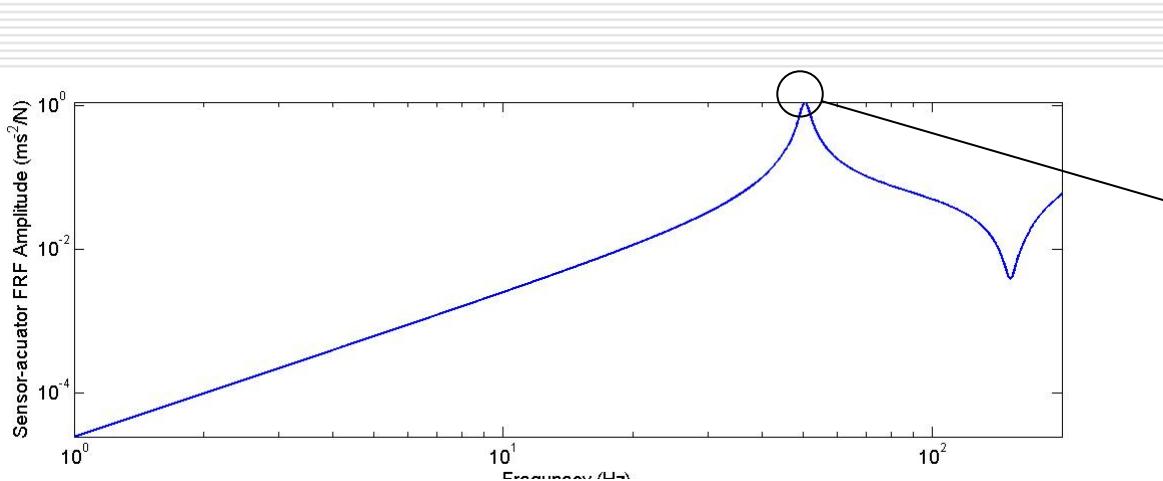
Adaptive configurations

- #1 Negative acceleration feedback with small PZT actuators and linear acceleration sensors



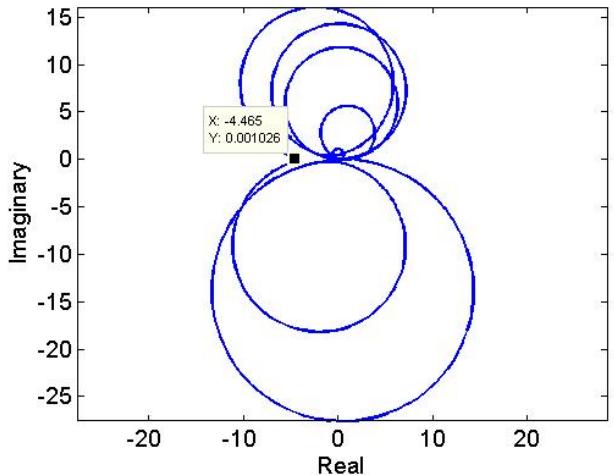
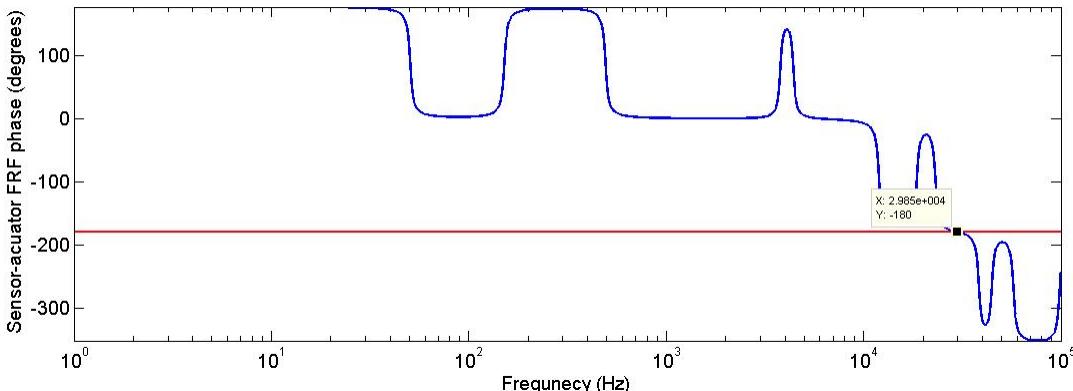
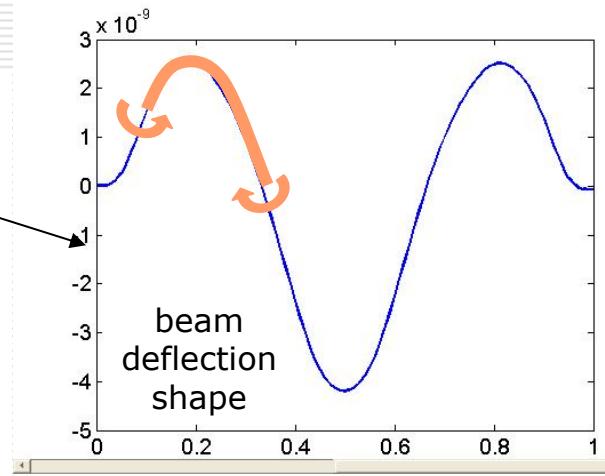
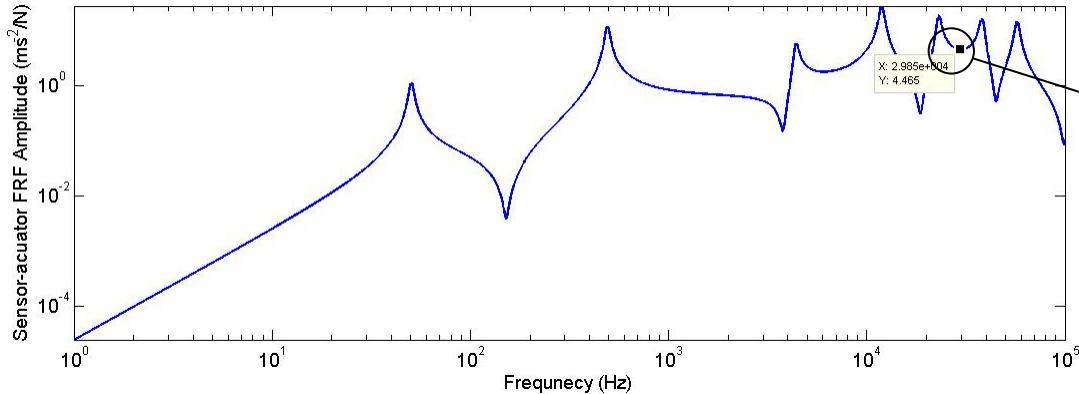
Adaptive configuration #1: Stability

- Sensor actuator open loop frequency response function 0-200 Hz



Adaptive configuration #1: Stability

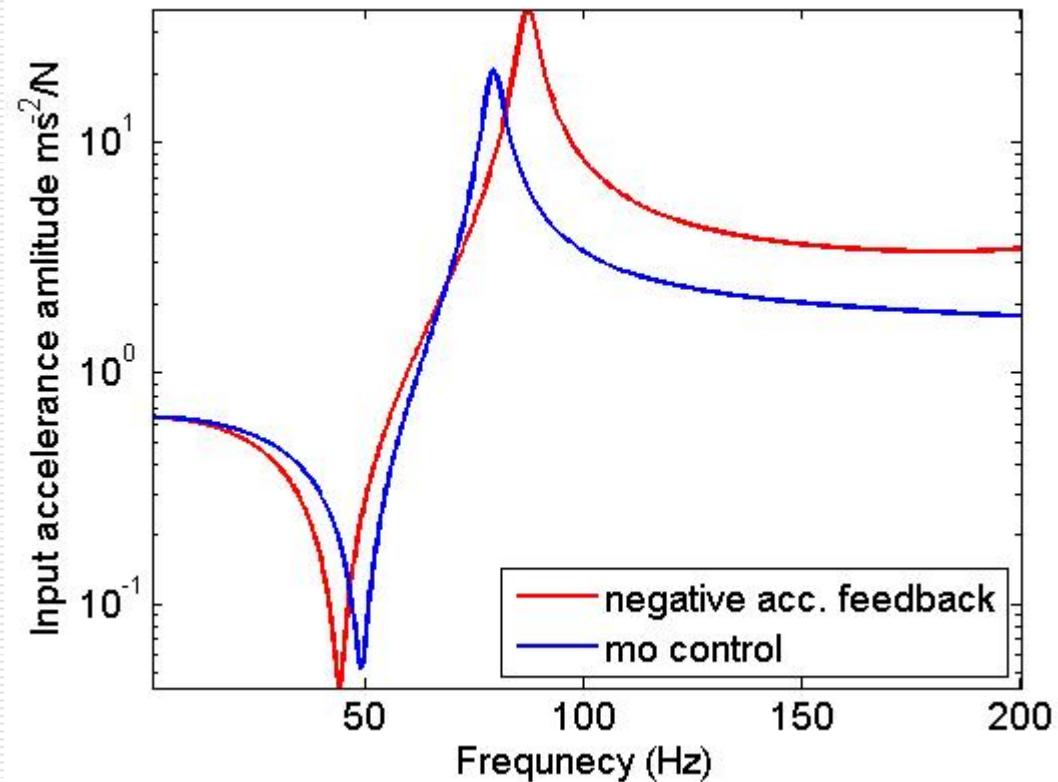
- Sensor actuator open loop frequency response function 0-100 kHz



$g_{\max} = 0.22 \text{ Ns}^2$

Adaptive configuration #1: Adaptability

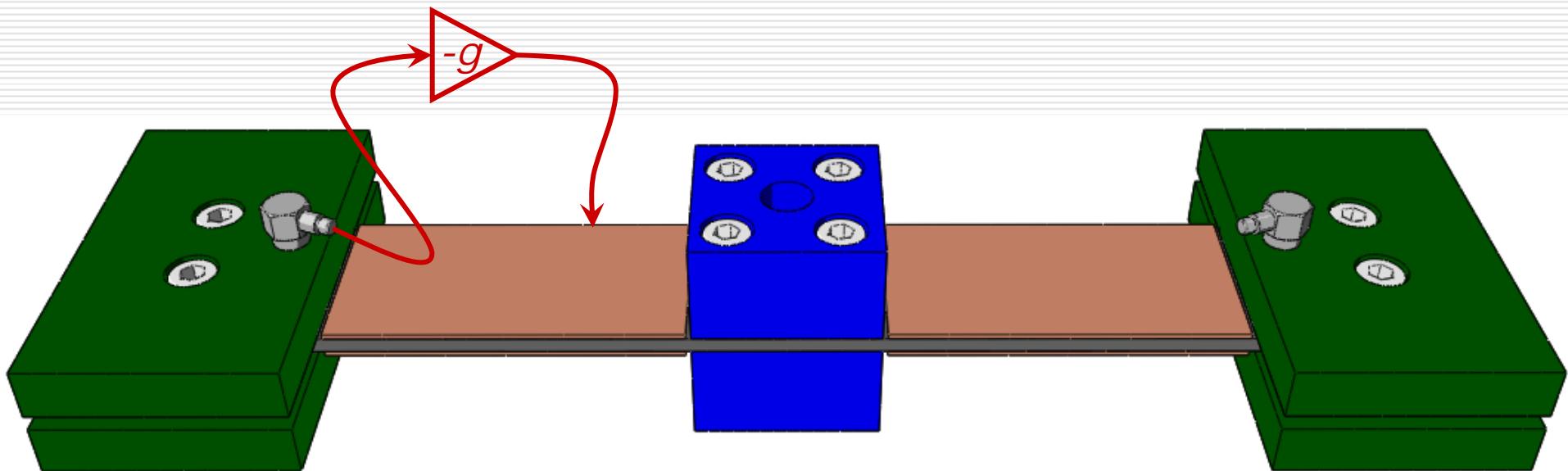
- Input acceleration 0-200 Hz



$g=0.05 \text{ Ns}^2$

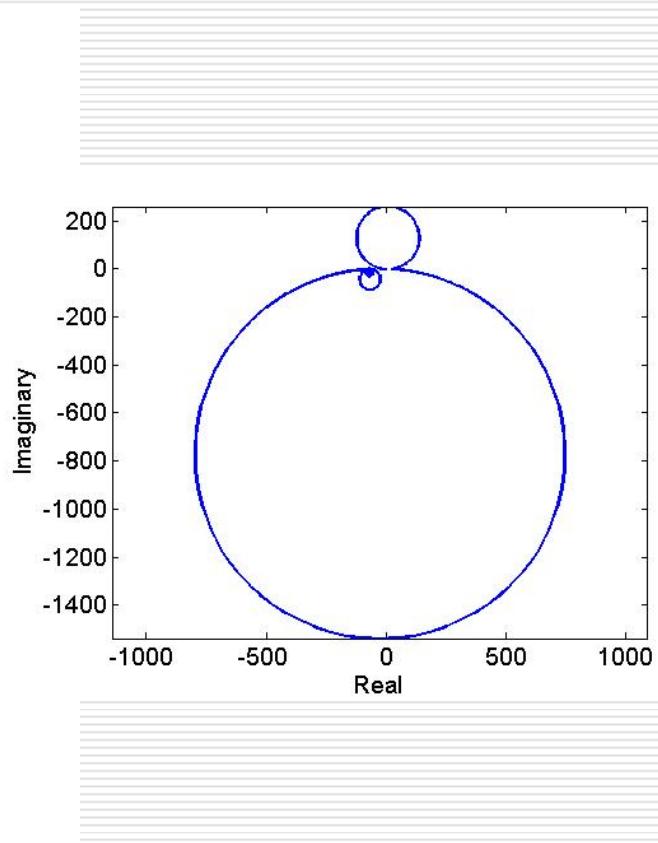
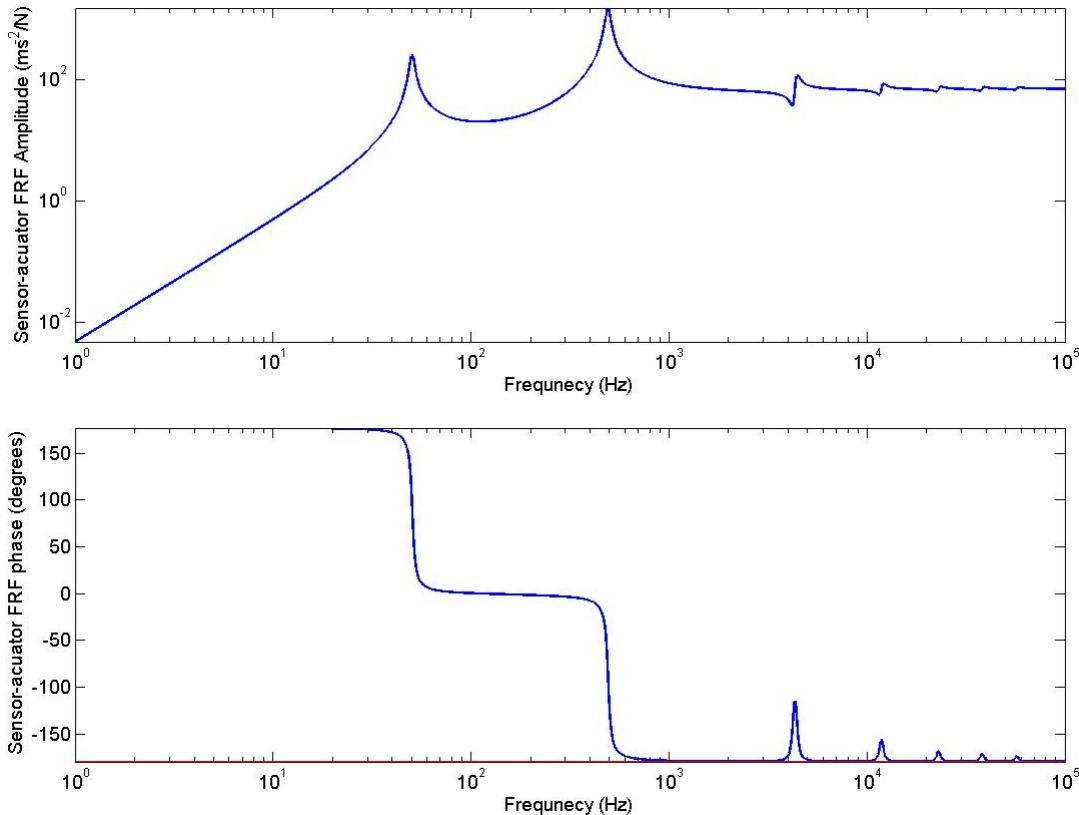
Adaptive configurations

- #2 Negative acceleration feedback with large PZT actuators and linear acceleration sensors



Adaptive configuration #2: Stability

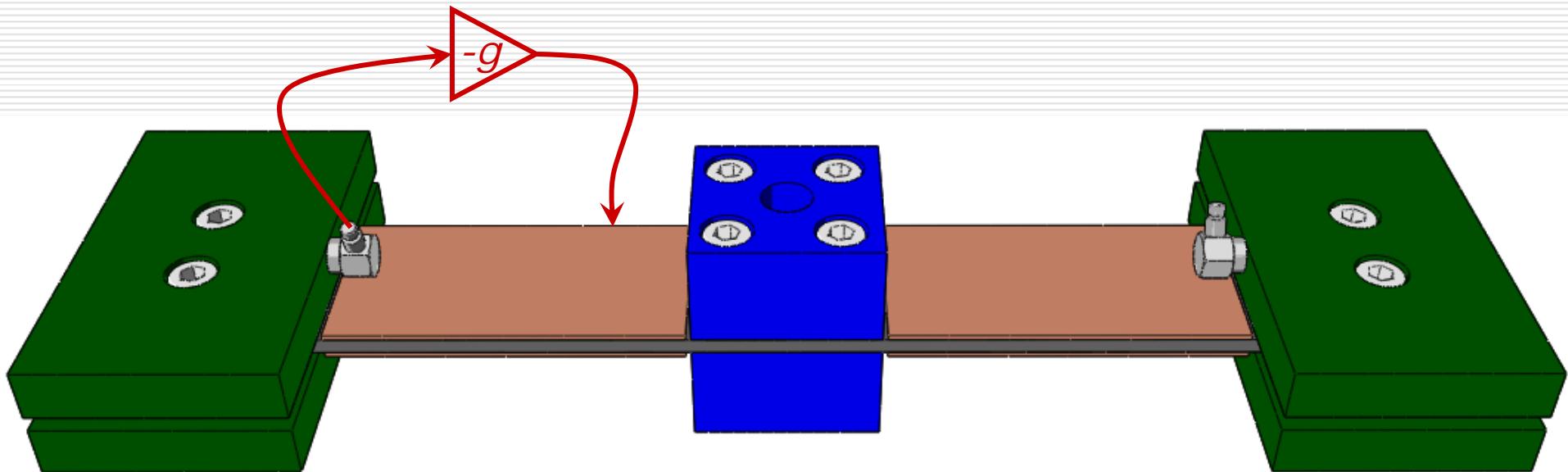
- Sensor actuator open loop frequency response function 0-100 kHz



$g_{\max} = \infty$, phase margin low

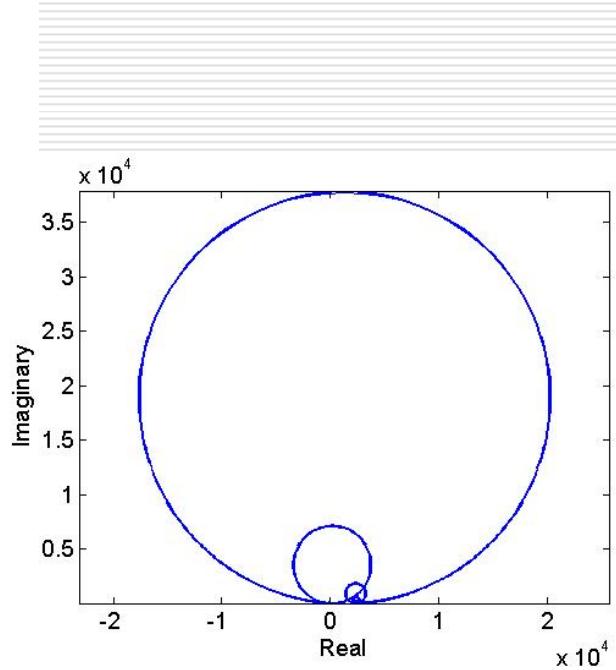
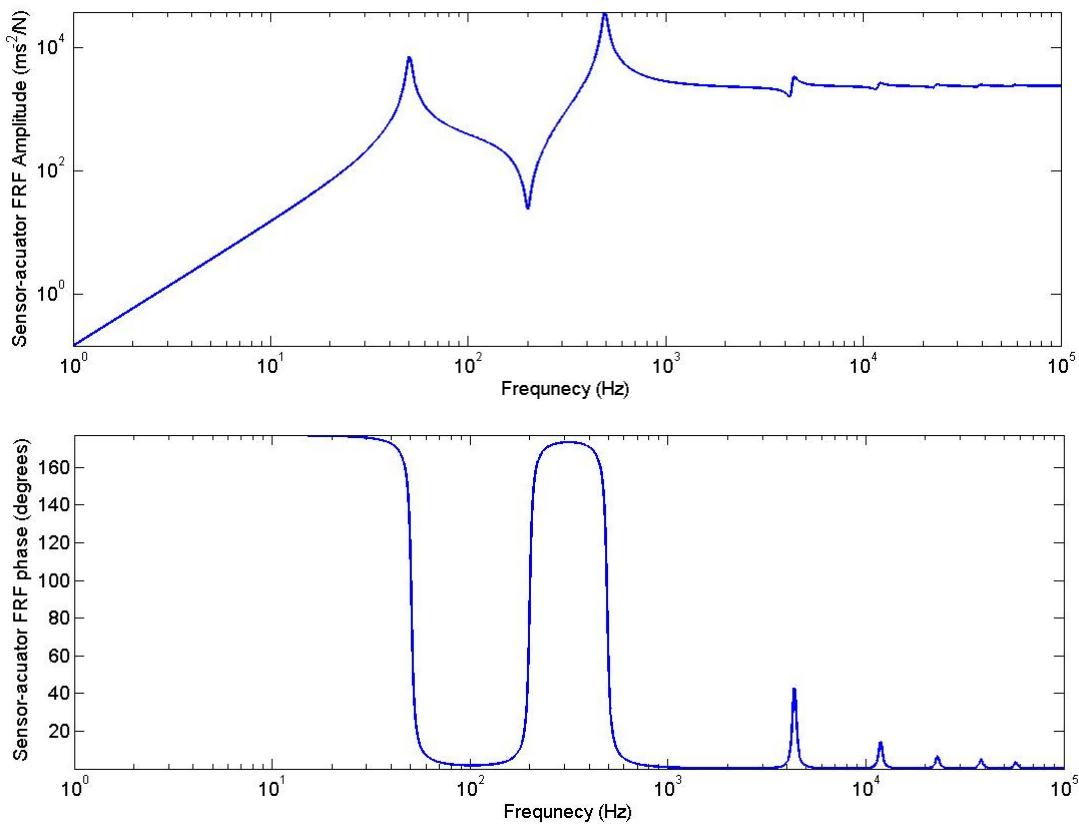
Adaptive configurations

- #3 Negative acceleration feedback with large PZT actuators and angular acceleration sensors



Adaptive configuration #3: Stability

- Sensor actuator open loop frequency response function 0-100 kHz

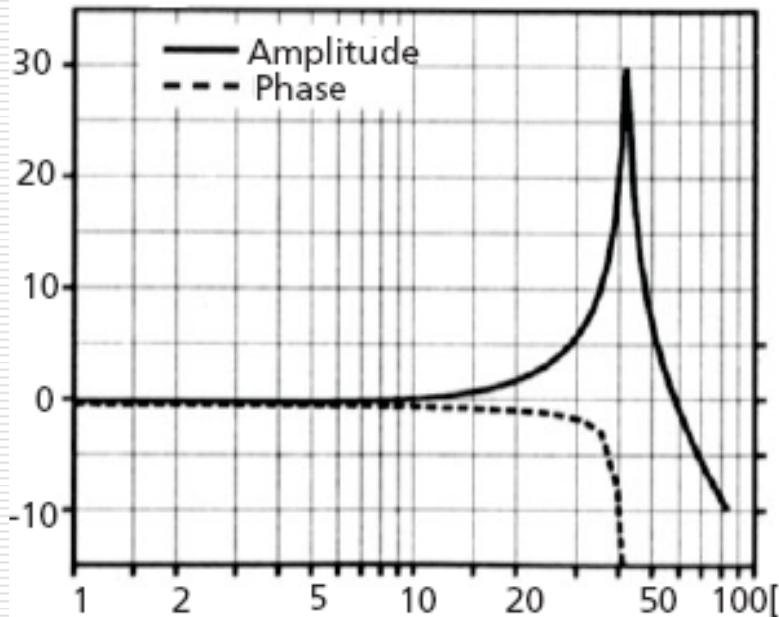


$g_{\max} = \infty$, better phase margin

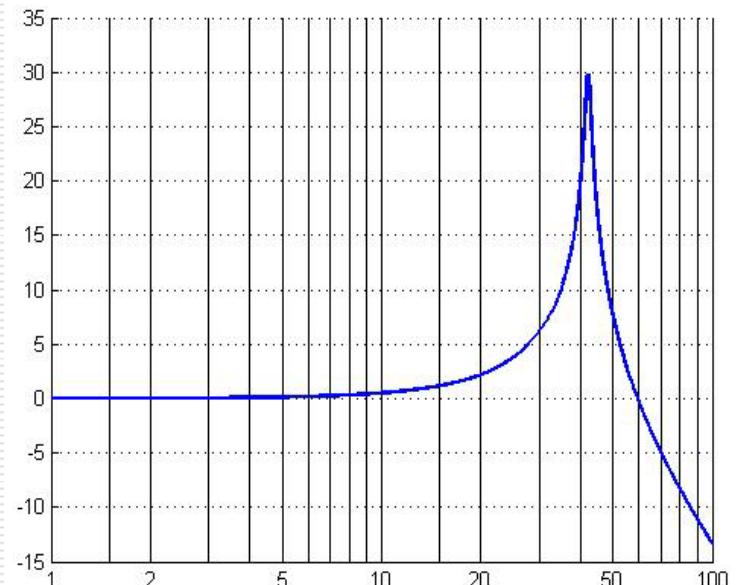
Adaptive configuration #3: Stability

□ But what about the sensor dynamics?

B&K 4371 calibration chart

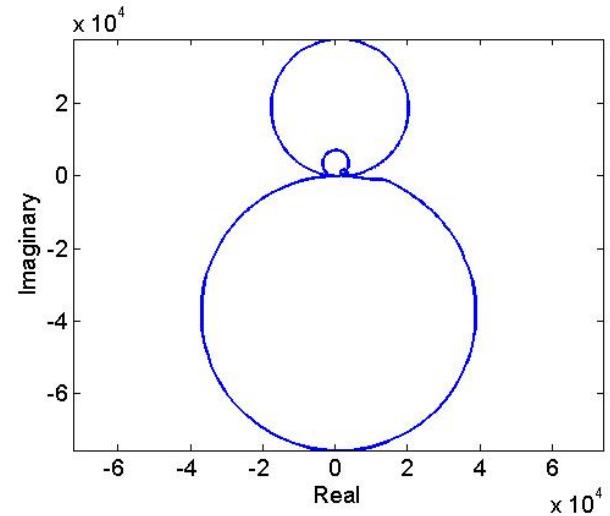
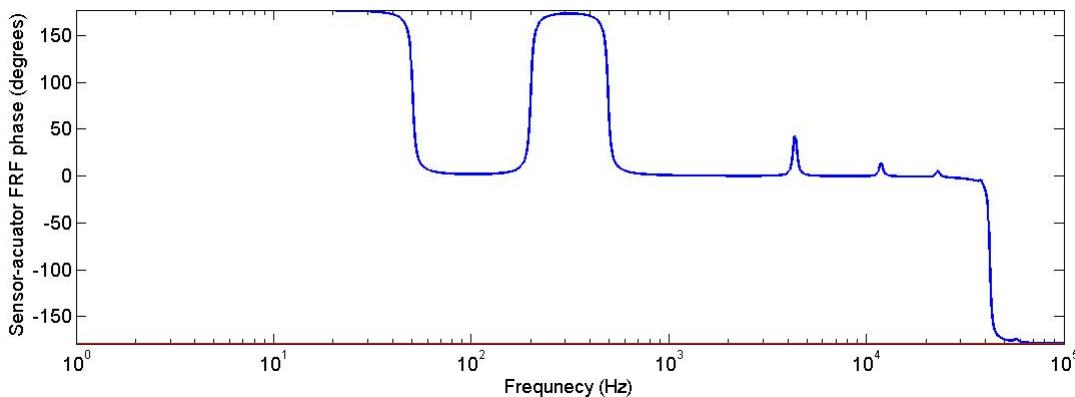
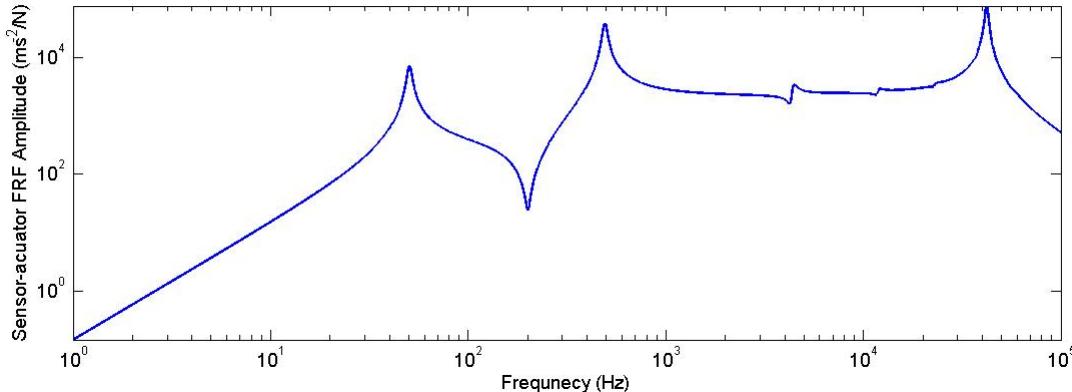


1 DOF model



Adaptive configuration #3: Stability

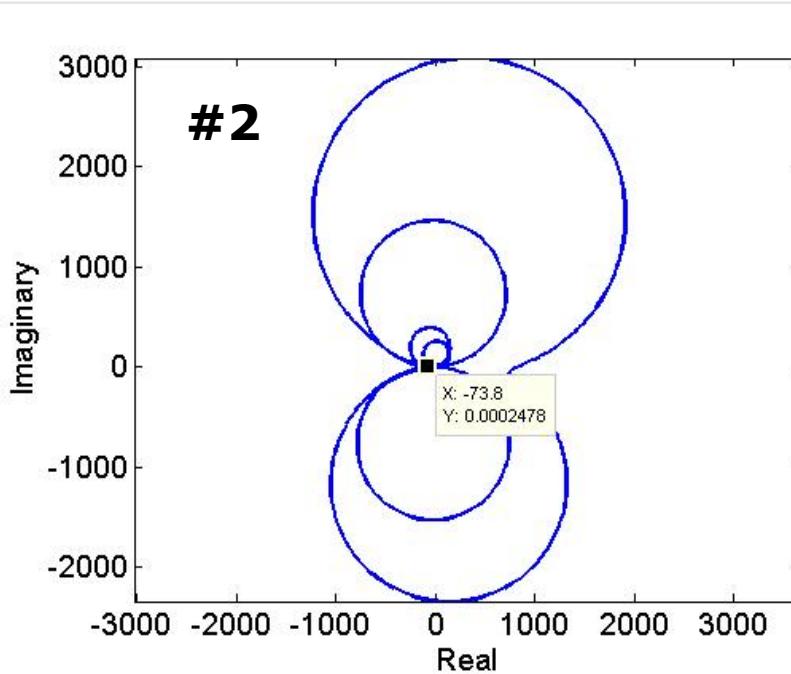
- Sensor actuator open loop frequency response function 0-100 kHz including the sensor frequency response



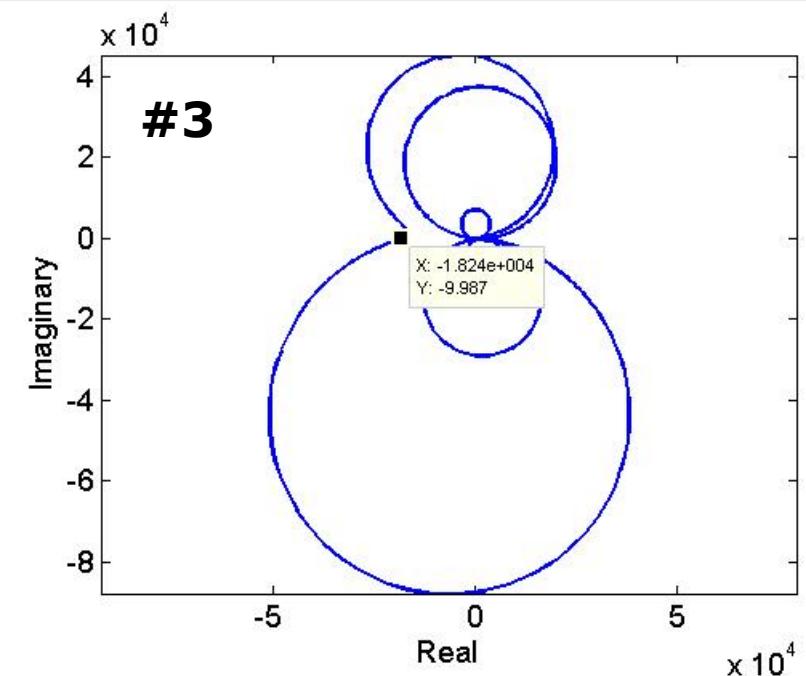
$g_{\max} = \infty \Rightarrow$ sensor response does not appear to disrupt the stability BUT ...

Adaptive configuration #2 & #3: Stability

- ...what if there is a small mismatch (3%) between the length of the piezo and the length of the beam?



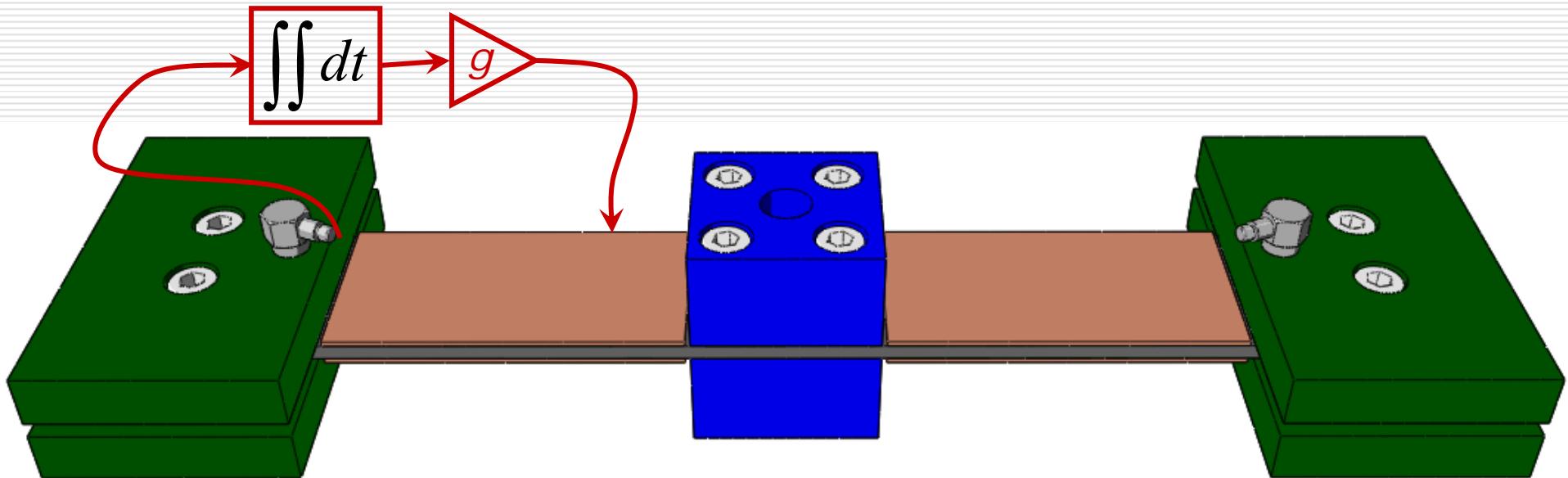
$$g_{\max} = 0.014 \text{ Ns}^2$$



$$g_{\max} \approx 0$$

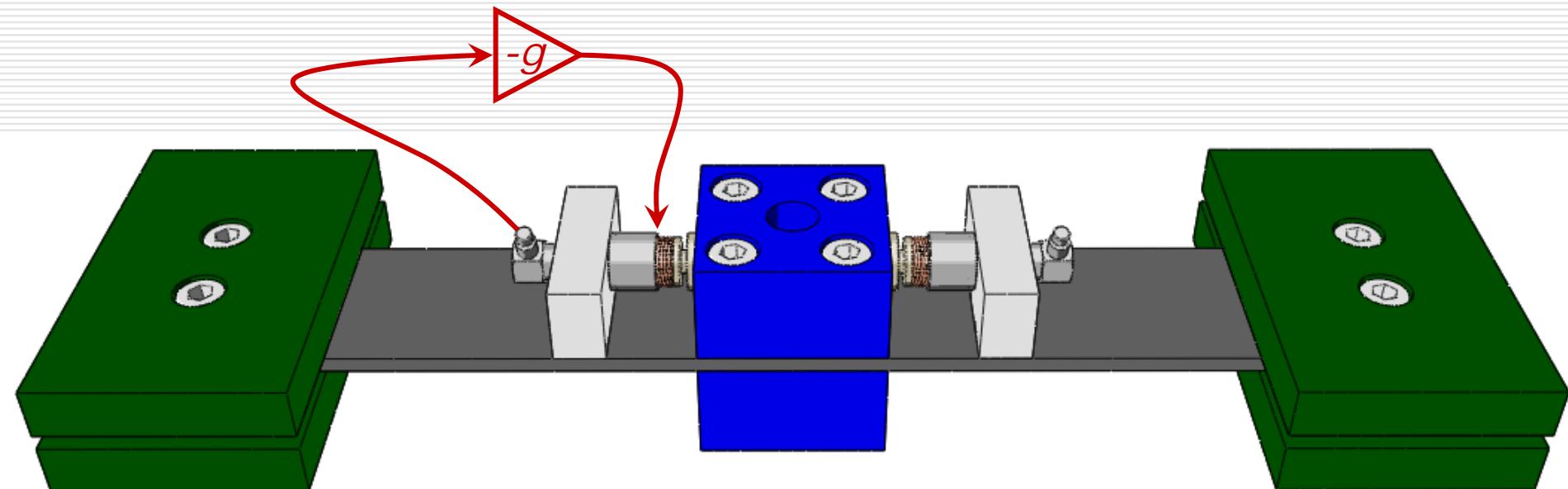
Future work: Adaptive configurations

- #4 Positive position feedback with large PZT actuators and linear displacement sensors



Future work: Adaptive configurations

- #5 negative acceleration feedback with voice coil actuators and linear acceleration sensors



Future work: Adaptive configurations

- #6 Positive position feedback with small PZT deformation angle sensors and linear force actuators

