

Generating Parametric Models from Tabulated Data

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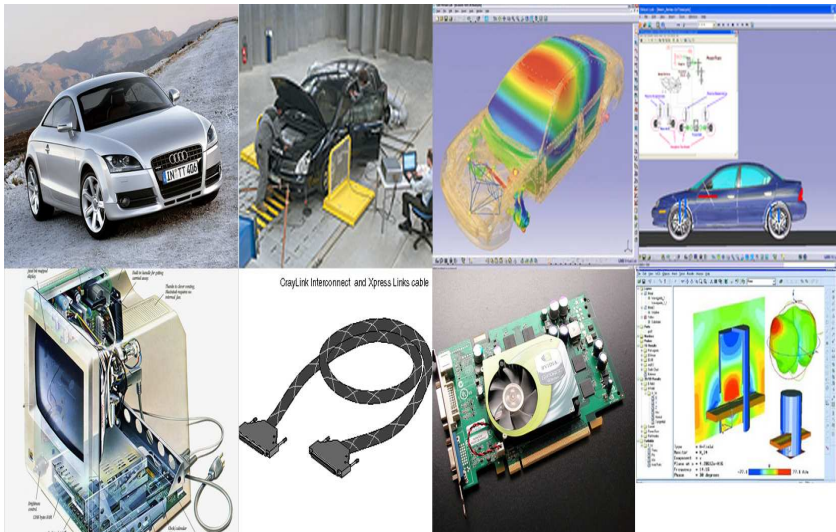
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Outline

- 1 *Motivation*
- 2 *Problem Statement*
- 3 *Unitary Constraint*
- 4 *Choice of s_0*
- 5 *Stability*
- 6 *Results*
- 7 *Conclusion*

Motivation



Previous Work

Currently available approaches:

- vector fitting (VF) used to construct models for known parameter values, followed by a parametrization of the numerator and denominator of the transfer function by linear combinations of basis functions which are piecewise linear in each parameter [9, 8].
- generalization of the Sanathanan-Koerner (SK) iteration (VF is a particular case of SK) to the parametric case [10, 7]
- multivariate formulation of the Orthonormal Vector Fitting technique [2, 1]
- recursive algorithm to compute the parametrized residues of the multivariate transfer function [3]
- generalization of multivariate Vector Fitting which includes parameter derivatives [4].

are **time & memory consuming**.

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Parametric modeling from measurements

Σ in descriptor-form: $\mathbf{E}\dot{\mathbf{x}}(t) = \mathbf{A}\mathbf{x}(t) + \mathbf{B}\mathbf{u}(t)$, $\mathbf{y}(t) = \mathbf{C}\mathbf{x}(t) + \mathbf{D}\mathbf{u}(t)$,
 $\mathbf{x}(t)$: state, $\mathbf{u}(t)$: input, $\mathbf{y}(t)$: corresponding output, $\mathbf{E}, \mathbf{A} \in \mathbb{R}^{n \times n}$, $\mathbf{B} \in \mathbb{R}^{n \times p}$,
 $\mathbf{C} \in \mathbb{R}^{p \times n}$, $\mathbf{D} \in \mathbb{R}^{p \times p}$ are constant; $[\mathbf{E}, \mathbf{A}, \mathbf{B}, \mathbf{C}, \mathbf{D}]$: a realization of Σ .

- tabulated/measurement data (e.g. S-parameters) given wrt frequencies f_i , $i=1, \dots, n_f$, but also wrt one or more parameters as:

$$\left(f_i, \alpha_k, \mathbf{S}^{(i,k)} := \begin{bmatrix} S_{11}^{(i,k)} & \dots & S_{1p}^{(i,k)} \\ \vdots & \vdots & \vdots \\ S_{p1}^{(i,k)} & \dots & S_{pp}^{(i,k)} \end{bmatrix} \right) \quad (1)$$

- one parameter α taking values α_k , $k=1, \dots, n_\alpha$

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- one parameter α taking values α_k , $k=1, \dots, n_\alpha$
- Goal**: find a **parametric** system $\Sigma(\alpha)$ which models (1), st its transfer function computed for α_k , evaluated at $j \cdot 2\pi f_i$, is close to $\mathbf{S}^{(i,k)}$: $\mathbf{H}^{(\alpha_k)}(2\pi j f_i) = \mathbf{C}^{(\alpha_k)}(2\pi j f_i \mathbf{I} - \mathbf{A}^{(\alpha_k)})^{-1} \mathbf{B}^{(\alpha_k)} \approx \mathbf{S}^{(i,k)}$, $i=1, \dots, n_f$
- Moreover**, $\Sigma(\alpha)$ for other α , should be close to the model one would obtain from measurements performed for these new values.

What we propose

Modeling step: **Problem:** construct LTI models $\Sigma(\alpha_k)$ for some fixed parameter values α_k (rational approximation or system identification - can be solved in many ways)

Loewner matrix framework & Tangential interpolation [6, 5]

- fast, accurate & robust
- especially designed for many ports
- models of small dimension

Generating Parametric Models: Interpolate between state-space matrices, after applying a suitable similarity transformation

- choice between different interpolation schemes
- choice between the canonical form in which systems are brought initially
- constrain the transformation to be close to unitary

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Unitary Constraint

Motivation

Recall: We want to solve

$$J(\mathbf{T}) = \alpha \left\| \mathbf{M}^{(0)} - \mathbf{T}^{-1} \mathbf{M}^{(1)} \mathbf{T} \right\|^2 + \beta \left\| \mathbf{B}^{(0)} - \mathbf{T}^{-1} \mathbf{B}^{(1)} \right\|^2 + \gamma \left\| \mathbf{C}^{(0)} - \mathbf{C}^{(1)} \mathbf{T} \right\|^2 \quad (2)$$

but, instead, we solve

$$\tilde{J}(\mathbf{T}) = \alpha \left\| \mathbf{T} \mathbf{M}^{(0)} - \mathbf{M}^{(1)} \mathbf{T} \right\|^2 + \beta \left\| \mathbf{T} \mathbf{B}^{(0)} - \mathbf{B}^{(1)} \right\|^2 + \gamma \left\| \mathbf{C}^{(0)} - \mathbf{C}^{(1)} \mathbf{T} \right\|^2 \quad (3)$$

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They are **different**, but by introducing an additional constraint in (3) to enforce \mathbf{T} to be close to **unitary** (\mathbf{T} is unitary if $\mathbf{T}^* \mathbf{T} = \mathbf{T} \mathbf{T}^* = \mathbf{I}$), the approximation of (2) by (3) makes sense.

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Why?

$$\left\| \mathbf{T} \mathbf{M}^{(0)} - \mathbf{M}^{(1)} \mathbf{T} \right\|^2 = \left\| \mathbf{T}^{-1} \left(\mathbf{T} \mathbf{M}^{(0)} - \mathbf{M}^{(1)} \mathbf{T} \right) \right\|^2 = \left\| \mathbf{M}^{(0)} - \mathbf{T}^{-1} \mathbf{M}^{(1)} \mathbf{T} \right\|^2$$

because the 2-norm is invariant under a unitary change of basis.

Unitary Constraint

We add an additional term to (3)

$$\hat{J}(\mathbf{T}) = \alpha \left\| \mathbf{T}\mathbf{M}^{(0)} - \mathbf{M}^{(1)}\mathbf{T} \right\|^2 + \beta \left\| \mathbf{T}\mathbf{B}^{(0)} - \mathbf{B}^{(1)} \right\|^2 + \gamma \left\| \mathbf{C}^{(0)} - \mathbf{C}^{(1)}\mathbf{T} \right\|^2 + \delta \left\| \mathbf{T}^*\mathbf{T} - \mathbf{I} \right\|^2,$$

δ appropriate scaling factor. Forming the Fréchet derivative, we have

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$$\begin{aligned} & \left(\alpha \mathbf{M}^{(1)*} \mathbf{M}^{(1)} + \gamma \mathbf{C}^{(1)*} \mathbf{C}^{(1)} \right) \mathbf{T} + \mathbf{T} \left(\alpha \mathbf{M}^{(0)} \mathbf{M}^{(0)*} + \beta \mathbf{B}^{(0)} \mathbf{B}^{(0)*} \right) - \alpha \left(\mathbf{M}^{(1)} \mathbf{T} \mathbf{M}^{(0)*} + \mathbf{M}^{(1)*} \mathbf{T} \mathbf{M}^{(0)} \right) \\ & + \delta 4 \mathbf{T} (\mathbf{T}^* \mathbf{T} - \mathbf{I}) = \beta \mathbf{B}^{(1)} \mathbf{B}^{(0)*} + \gamma \mathbf{C}^{(1)*} \mathbf{C}^{(0)} \end{aligned} \quad (4)$$

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We add an additional term to (3)

$$\hat{J}(\mathbf{T}) = \alpha \left\| \mathbf{T}\mathbf{M}^{(0)} - \mathbf{M}^{(1)}\mathbf{T} \right\|^2 + \beta \left\| \mathbf{T}\mathbf{B}^{(0)} - \mathbf{B}^{(1)} \right\|^2 + \gamma \left\| \mathbf{C}^{(0)} - \mathbf{C}^{(1)}\mathbf{T} \right\|^2 + \delta \left\| \mathbf{T}^*\mathbf{T} - \mathbf{I} \right\|^2,$$

δ appropriate scaling factor. Forming the Fréchet derivative, we have

$$\begin{aligned} & \left(\alpha \mathbf{M}^{(1)*} \mathbf{M}^{(1)} + \gamma \mathbf{C}^{(1)*} \mathbf{C}^{(1)} \right) \mathbf{T} + \mathbf{T} \left(\alpha \mathbf{M}^{(0)} \mathbf{M}^{(0)*} + \beta \mathbf{B}^{(0)} \mathbf{B}^{(0)*} \right) - \alpha \left(\mathbf{M}^{(1)} \mathbf{T} \mathbf{M}^{(0)*} + \mathbf{M}^{(1)*} \mathbf{T} \mathbf{M}^{(0)} \right) \\ & + \delta 4 \mathbf{T} (\mathbf{T}^* \mathbf{T} - \mathbf{I}) = \beta \mathbf{B}^{(1)} \mathbf{B}^{(0)*} + \gamma \mathbf{C}^{(1)*} \mathbf{C}^{(0)} \end{aligned} \quad (4)$$

- $\mathbf{T}(\mathbf{T}^*\mathbf{T} - \mathbf{I})$ makes (4) nonlinear in \mathbf{T} , so it cannot be solved directly
- we use a Newton-like procedure on the linearized (4), obtained by writing \mathbf{T} as $\mathbf{T}_0 + \Delta\mathbf{T}$ and disregarding higher order terms in $\Delta\mathbf{T}$
- we start with the initial guess \mathbf{T}_0 and add, at each step, a correction $\Delta\mathbf{T}$ which will make the solution \mathbf{T} close to being unitary.

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Where to choose the test frequency?

Recall: $\Delta^{(\alpha)}(s_0) = (1-\alpha)\mathbf{H}^{(0)}(s_0) + \alpha\mathbf{H}^{(1)}(s_0) - \mathbf{H}^{(\alpha)}(s_0)$.

- chosen away from the system poles
- $s_0 \in \mathbb{R}$

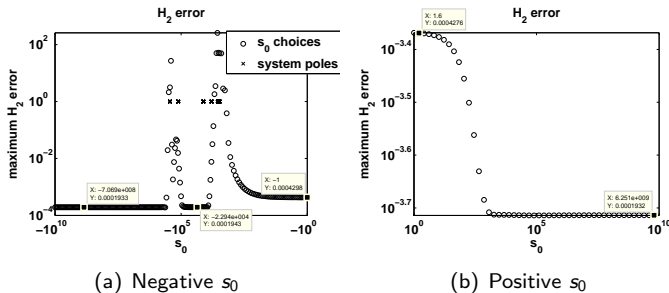


Figure: The influence of s_0 on the errors

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Stability

Fact: Systems built for known parameter configurations are stable.

Question: Will the system obtained with our procedure also be stable for any new parameter value?

Answer: Lyapunov stability theory [11].

Definition

Let L be a continuous map from \mathbb{R}^n to \mathbb{R} . It is called a **Lyapunov function** for system $\dot{\mathbf{x}}(t)=f(\mathbf{x}(t))$ if:

- L is locally positive definite ($L(\mathbf{x})>0$, $0<\|\mathbf{x}\|<r_1$, for some r_1) &
- \dot{L} is locally negative semidefinite ($\dot{L}(\mathbf{x})\leq 0$, $0<\|\mathbf{x}\|<r_2$).

Theorem

- If $\exists L(\mathbf{x})$ for system $\dot{\mathbf{x}}(t)=f(\mathbf{x}(t))$, then $\mathbf{x}=\mathbf{0}$ is a **stable equilibrium point in the sense of Lyapunov**.
- If $\dot{L}(\mathbf{x})<0$, $0<\|\mathbf{x}\|<r_2$, for some r_2 , then $\mathbf{x}=\mathbf{0}$ is an **asymptotically stable equilibrium point**.

Stability

Define a Lyapunov candidate function $L(\mathbf{x}) = \mathbf{x}^* \mathbf{P} \mathbf{x}$, where \mathbf{P} is symmetric positive definite.

$$\dot{L}(\mathbf{x}) = \dot{\mathbf{x}}^* \mathbf{P} \mathbf{x} + \mathbf{x}^* \mathbf{P} \dot{\mathbf{x}} = \mathbf{x}^* \mathbf{A}^* \mathbf{P} \mathbf{x} + \mathbf{x}^* \mathbf{P} \mathbf{A} \mathbf{x} = \mathbf{x}^* \underbrace{(\mathbf{A}^* \mathbf{P} + \mathbf{P} \mathbf{A})}_{-\mathbf{Q}} \mathbf{x} < 0 \quad (5)$$

- if $\mathbf{Q} \geq \mathbf{0}$, $\mathbf{x} = \mathbf{0}$ is a **stable equilibrium point**.
- if $\mathbf{Q} > \mathbf{0}$, $\mathbf{x} = \mathbf{0}$ is **globally asymptotically stable** & **the system is stable**.
- this can be expressed as a *linear matrix inequality* (LMI):
 $\mathbf{P} \mathbf{A} + \mathbf{A}^* \mathbf{P} < \mathbf{0}$ which always has a solution for \mathbf{A} stable.

Stability

$\Sigma(\alpha_k)$ are stable, for $k=1, \dots, n_\alpha$, and for a new α ,

$$\mathbf{A}^{(\alpha)} = \sum_{k=1}^{n_\alpha} w_k \mathbf{A}^{(\alpha_k)}, \quad \mathbf{B}^{(\alpha)} = \sum_{k=1}^{n_\alpha} w_k \mathbf{B}^{(\alpha_k)} \quad (6)$$

and similarly for $\mathbf{C}^{(\alpha)}$, $\mathbf{D}^{(\alpha)}$, with $\sum_{k=1}^{n_\alpha} w_k = 1$.

Goal: Find a **common** solution \mathbf{P} , $\forall k=1, \dots, n_\alpha$, to the inequalities

$$\mathbf{A}^{(\alpha_k)*} \mathbf{P} + \mathbf{P} \mathbf{A}^{(\alpha_k)} < \mathbf{0} \Leftrightarrow \mathbf{A}^* \text{diag}(\mathbf{P}, \dots, \mathbf{P}) + \text{diag}(\mathbf{P}, \dots, \mathbf{P}) \mathbf{A} < \mathbf{0} \quad (7)$$

$$w_k \mathbf{A}^{(\alpha_k)*} \mathbf{P} + \mathbf{P} w_k \mathbf{A}^{(\alpha_k)} < \mathbf{0} \Rightarrow \underbrace{\sum_{k=1}^{n_\alpha} w_k \mathbf{A}^{(\alpha_k)*} \mathbf{P} + \mathbf{P} \sum_{k=1}^{n_\alpha} w_k \mathbf{A}^{(\alpha_k)}}_{\mathbf{A}^{(\alpha)}} < \mathbf{0}$$

where $\mathbf{A} = \text{diag}(\mathbf{A}^{(\alpha_1)}, \dots, \mathbf{A}^{(\alpha_{n_\alpha})})$.

Solved with Matlab's LMI toolbox.

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Error Measures

- normalized \mathcal{H}_∞ -norm of the error system

$$\mathcal{H}_\infty \text{ error} = \frac{\max_{i=1\dots k} \sigma_1(\mathbf{H}(j\omega_i) - \mathbf{S}^{(i)})}{\max_{i=1\dots k} \sigma_1(\mathbf{S}^{(i)})},$$

- normalized \mathcal{H}_2 -norm of the error system

$$\mathcal{H}_2 \text{ error} = \frac{\sum_{i=1}^k \|\mathbf{H}(j\omega_i) - \mathbf{S}^{(i)}\|_F^2}{\sum_{i=1}^k \|\mathbf{S}^{(i)}\|_F^2}.$$

where

$$\|\mathbf{H}(j\omega_i) - \mathbf{S}^{(i)}\|_F^2 = \sum_{k_1=1}^p \sum_{k_2=1}^p \left| \mathbf{H}_{k_1, k_2}(j\omega_i) - \mathbf{S}_{k_1, k_2}^{(i)} \right|^2.$$

Microstrip lines and an RC pair

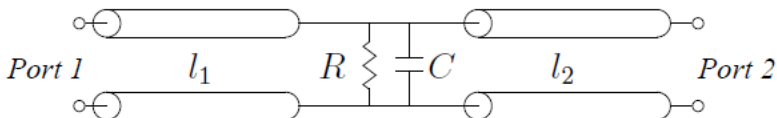


Figure: Microstrip lines and an RC pair

Nominal values: $R = 4\text{k}\Omega$, $C = 0.2\text{pF}$, $w = 80\mu\text{m}$, $l_1 = 3\text{cm}$, $l_2 = 2\text{cm}$, for width & lengths of the microstrips, and $h = 0.3\text{mm}$, $\epsilon_r = 4$, for the dielectric height & permittivity.

This reproduces an interconnect link loaded by a device.

Design parameter: width w

- S-parameters of this 2-port system computed (via a full wave simulation) for 100 frequencies between 10MHz and 10GHz
- 15 values of w between 60 and $130\mu\text{m}$ in steps of $5\mu\text{m}$
- for better conditioning, frequencies were scaled by 10^{-6}
- we use 8 responses for $w=\{60, 70, \dots, 120, 130\} \mu\text{m}$ for modeling
- we identified systems of order 20 with $\mathbf{D} = \mathbf{0}$
- CPU time for one model was 0.03s on average, and all 8 took 0.25s
- LMI (7) was solved for a common $\mathbf{P} \Rightarrow$ all our parametric models will be stable (no matter how the weights are chosen)
- $w = 90\mu\text{m}$ was chosen as the reference system
- for all remaining systems, \mathbf{T} in (3) was applied as a similarity transformation; this took 0.51s

Different canonical forms and interpolation schemes

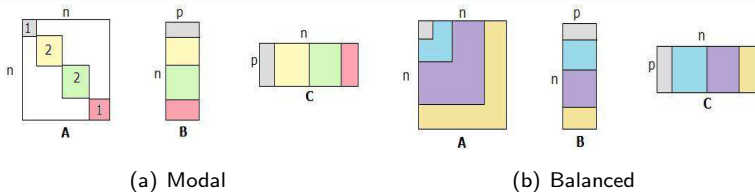


Figure: Different canonical forms

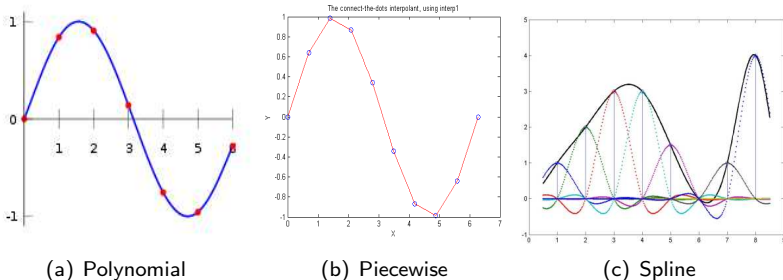


Figure: Plots for different interpolation schemes

Results for design parameter: width w

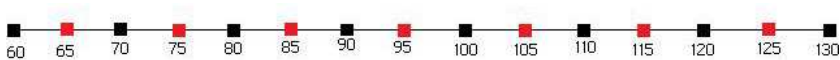


Figure: Values for w

	\mathcal{H}_2 max	\mathcal{H}_2 min	\mathcal{H}_∞ max	\mathcal{H}_∞ min
modeling	1.3309e-4	1.1719e-4	1.2084e-3	1.1030e-3
polynomial with balanced	1.9319e-4	1.1939e-4	1.2491e-3	1.0970e-3
polynomial with modal	7.3754e-3	5.3365e-4	2.2093e-2	1.4944e-3
piecewise with balanced	5.8575e-4	2.4685e-4	1.3116e-3	1.1477e-3
piecewise with modal	5.4896e-3	8.2878e-4	1.3753e-2	2.2075e-3
spline with balanced	1.4855e-4	1.1979e-4	1.2202e-3	1.1022e-3
spline with modal	5.2803e-3	1.5953e-4	1.2509e-2	1.1517e-3
validation	1.3213e-4	1.1860e-4	1.2032e-3	1.1140e-3

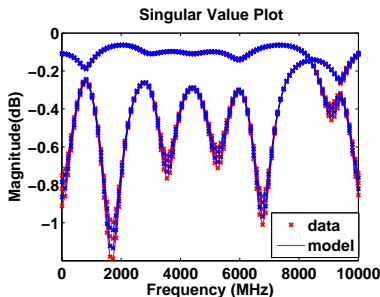
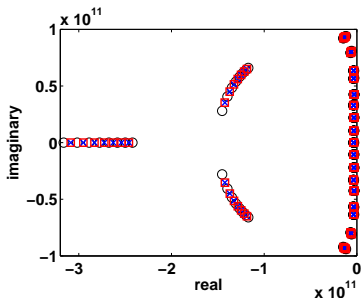
Table: Errors

*Results for unitary constraint for design parameter:
width w*

	\mathcal{H}_2 max	\mathcal{H}_2 min	\mathcal{H}_∞ max	\mathcal{H}_∞ min
polynomial with balanced	1.9236e-4	1.1938e-4	1.2483e-3	1.0962e-3
polynomial with modal	1.3852	1.6774e-1	4.6274	3.6683e-1
piecewise with balanced	6.4347e-4	3.0443e-4	1.2204e-3	1.0917e-3
piecewise with modal	7.8957e-1	3.7693e-3	2.4783	1.4491e-2
spline with balanced	1.4854e-4	1.1956e-4	1.2202e-3	1.1022e-3
spline with modal	9.0187e-1	6.1852e-2	2.7540	1.1561e-1

Table: Errors when adding the unitary constraint

Plots for design parameter: width w



(a) Evolution of poles wrt parameters (b) Singular values of the S-parameter magnitude (black circles: poles of the systems used for parameter modeling, blue crosses: poles of the parametric systems, red squares: true poles of the validation systems)

Figure: Plots for design parameter: width w

Results for design parameters: resistor R & capacitor C

	\mathcal{H}_2 max	\mathcal{H}_2 min	\mathcal{H}_∞ max	\mathcal{H}_∞ min
modeling	1.9043e-4	1.7928e-4	1.8470e-3	1.7432e-3
polynomial with balanced	3.0840e-4	1.7929e-4	1.9073e-3	1.7233e-3
polynomial with modal	1.6616e-2	1.7894e-4	6.1901e-2	1.7532e-3
piecewise with balanced	1.5603e-3	1.7927e-4	5.5619e-3	1.7090e-3
piecewise with modal	1.6616e-2	1.7927e-4	6.1901e-2	1.7540e-3
spline with balanced	3.0840e-4	1.7929e-4	1.9073e-3	1.7233e-3
spline with modal	1.6616e-2	1.7894e-4	6.1901e-2	1.7532e-3
validation	1.8938e-4	1.7941e-4	1.8481e-3	1.7531e-3

Table: Errors

Results for unitary constraint for design parameters: resistor R & capacitor C

	\mathcal{H}_2 max	\mathcal{H}_2 min	\mathcal{H}_∞ max	\mathcal{H}_∞ min
polynomial with balanced	1.4544e-3	1.7929e-4	6.8113e-3	1.7171e-3
polynomial with modal	4.0447	2.9460e-4	2.9092e+1	1.8190e-3
piecewise with balanced	7.3779e-3	1.7926e-4	3.8669e-2	1.6939e-3
piecewise with modal	3.7449	2.4148e-4	8.1928	1.8347e-3
spline with balanced	1.4544e-3	1.7929e-4	6.8113e-3	1.7171e-3
spline with modal	4.0447	2.9460e-4	2.9092e+1	1.8190e-3

Table: Errors when adding the unitary constraint

Plots for design parameters: resistor R & capacitor C

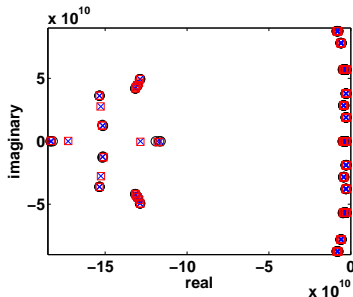
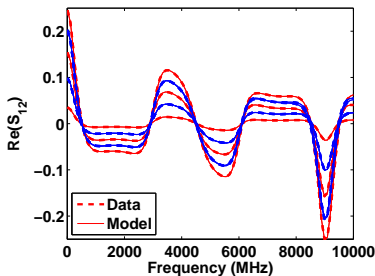
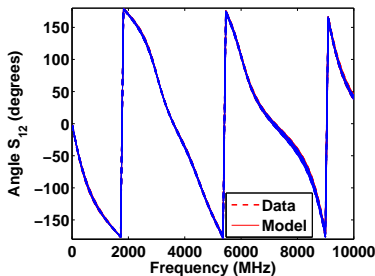


Figure: Evolution of poles wrt parameters (black circles: poles of the systems used for modeling, blue crosses: poles of the parametric systems, red squares: true poles of the validation systems)

Plots for design parameters: resistor R & capacitor C



(a) $\text{Re}(S_{12})$



(b) Angle S_{21}

Figure: Entries for some parameter values (red: systems used for modeling, blue: parametric systems)

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Conclusion

Results depend on

- initial canonical form:
 - modal form suitable for purely mechanical systems
 - balanced form suitable for systems with high damping
- interpolation scheme
 - piecewise is cheap, with not so good results
 - polynomial may lose accuracy when too many points are used
 - spline is accurate, but expensive (with Matlab's spline toolbox)

Thank you!

Questions?

References



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